

HEAT TRANSFER COEFFICIENTS
OF STEAM WATER MIXTURES



LEE WILSON FISHER
AND
JOHN MARSHALL KING

1953

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HEAT TRANSFER COEFFICIENTS
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Wiley
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STEAM WATER MIXTURES

by

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PREFACE

It has become apparent during the course of instruction at the Naval Postgraduate School that the phenomena of heat transfer is of ever increasing importance in many fields of engineering. The attainment of higher thermal efficiencies ultimately becomes a problem in heat transfer. The concept of the heat transfer coefficient has become an accepted method for predicting the relative ease with which heat flow will occur.

In conversations with Professor E. E. Drucker, of the U. S. Naval Postgraduate School, the authors were made aware of the lack of adequate data on transfer coefficients as they relate to wet steam. Such lack of data was considered sufficiently important to be investigated.

The experimental work of this thesis was conducted at the U. S. Naval Postgraduate School from March to June of 1953.

The authors desire to express their appreciation to Professor E. E. Drucker for his extensive assistance in the preparation of this thesis, to the Research Division of the Babcock and Wilcox Company for their invaluable aid in the instrumentation of the test section, and also to Chief Jones, USN, and Chief Wallace, USN, for their aid in the construction of the experimental set-up.

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TABLE OF SYMBOLS AND ABBREVIATIONS

A	Area of heat transfer surface, square feet.
c_p	Specific heat at constant pressure, Btu/(lb)(deg F).
d	Diameter, feet.
G	Mass velocity, lb/(hr)(sq ft of cross section).
h	Coefficient of heat transfer between fluid and surface, Btu/(hr)(sq ft)(deg F).
k	Coefficient of thermal conductivity, Btu/(hr)(ft)(deg F).
L	Heated length of tube, feet.
M	Mass flow rate, lb/hr.
μ	Fluid viscosity, lb/(hr)(ft).
P	Total pressure, lb/sq in.
π	3.1416....
ρ	Fluid density, lb/cu ft.
q	Rate of heat transfer, Btu/hr.
r	Radius, feet.
T	Temperature, degrees Fahrenheit.
v	Fluid velocity, ft/hr
x	Moisture content of steam, lb water/lb mixture, percent.
Nu	$\frac{hd}{k}$, Nusselt Group, dimensionless.
Pr	$\frac{\mu c_p}{k}$, Prandtl Group, dimensionless.
Re	$\frac{\rho V d}{\mu}$, Reynolds, Group, dimensionless.

Subscripts:

1	inlet.	w	water.	t	tube.	d	depth
2	outlet.	s	steam.	o	outer.	i	inner

SUMMARY

The object of the development and preparation of this thesis was to determine the local or point coefficients of heat transfer to wet steam at moderate pressures, and to note their manner of variation with respect to moisture content of the steam. To accomplish this, steam at a known condition was passed through a vertical steel tube of 0.50 inches inside diameter which was heated electrically.

Coefficients were determined under the following conditions: moisture content in the range from 0 to 6 percent, absolute pressures up to 215 psia, and a variation in mass velocity from 73,000 to 172,000 pounds per hour per square foot.

The effects of moisture content were found to be significant, causing a rapid increase in the coefficient with increasing percent moisture. The local coefficients as they are herein presented vary from 110 BTU's/hr-Ft²-°F for saturated steam to values approaching 6500 BTU's/hr-Ft²-°F for steam with a moisture content of about 6%.

INTRODUCTION

Steam power has been long established as the primary means of ship propulsion in the U. S. Navy. In the advent of atomic power for naval vessels, it is entirely probable that steam will continue to be the means by which this power is transmitted to the main engines.

The Navy's interest in the improvement of the thermal efficiencies of boilers has led to the utilization of higher temperatures and pressures. It is felt that additional improvement may be brought about by critical re-evaluation of the current concept of boiler design as it relates to heat transfer. Consequently, it is important to know as much as possible about the mechanism of heat flow to the working medium as it passes through successive stages from saturated water to saturated steam. One aspect of this problem may be solved by experimentally determining the coefficients of heat transfer to steam with varying degrees of moisture content.

In this thesis work, experimentation was conducted in an effort to determine the coefficients for steam with a quality of 96 to 100 percent. Although this is a fairly narrow range of moisture content, the results did tend to indicate the trend with which the transfer coefficients varied with quality. Time was a limiting factor in preventing further investigation of steam with higher moisture content.

Heat transfer coefficients of this kind, that is, coefficients relating to forced convection, are usually correlated with respect to the local Nusselt, Reynolds, and Prandtl numbers. Such correlation, however, is invariably made for fluids which are of a single phase only. Obviously this was not the case in the present investigation. In addition, a thorough survey of all existing literature failed to yield any data on values of density, viscosity, or thermal conductivity of wet steam, thus preventing the computation of the above dimensionless quantities. Further research to determine some of these parameters would be extremely useful.

This work may be considered as an extension of the investigation of heat transfer in two phase flow. It represents some of the initial data available for steam at relatively low flow rates and low heat flux densities.

CHAPTER I

REVIEW OF LITERATURE ON TWO PHASE FLOW

A survey of the broad field of heat transfer indicates that a large quantity of data has been collected on the heat flow to fluids inside tubes. Unfortunately very little data has been published with respect to gas-liquid mixtures.

The lack of information on temperature and velocity distributions has made it necessary to correlate experimental data on heat transfer coefficients by means of empirical equations in terms of certain dimensionless groups. Perhaps the most highly utilized equation is the Nusselt relation for single phase flow in forced convection through a tube (4):---

$$Nu=0.024 Re^{0.8} Pr^{0.4} \quad (1)$$

It has been found by McAdams and others (4), (5) from data for vaporization inside tubes that the local film coefficients, in the preheat section where the fluid is initially heated, are usually somewhat higher than would be predicted from the Nusselt relation. This result is logical from the standpoint that some vapor bubbles may have formed in the preheating section, with the possibility that they disrupt the film locally as they are formed on the tube wall. As they move out into the fluid they would condense thus releasing their heat of condensation. The combined effect is an increase in heat transfer coefficient. In a comparable manner, it might be expected that as far as the influence on

the film coefficient is concerned, the coefficient will increase when gas passes through a system simultaneously with a liquid.

In 1949 Bergelin(2) observed that if a gas and a liquid pass simultaneously in upward flow through a tube, as the gas-liquid ratio is increased, the following three flow conditions could be distinguished:

(a) bubble flow, when gas bubbles pass individually through the tube;

(b) slug flow, when alternatively slugs of gas and liquid pass through the tube, and

(c) annular flow, when the liquid flows in an annulus along the tube wall, while the gas passes at a much higher velocity through the center of the tube. Verschoor and Stemerding(8) in their investigation of the transfer coefficients to air and water mixtures verified the observations of Bergelin. They found that as their air to water mixture was increased, the heat transfer coefficient increased until a maximum was reached that coincided with the transition from slug to annular flow. Prior to reaching this maximum there was a change in slope in their curve which corresponded to the change from bubble flow to slug flow. It is interesting to note that their heat transfer coefficients increased with an increase in flow density, however this was of secondary importance compared to the influence of varying the air-water mixture.

Yoder and Dodge⁽⁹⁾ measured film coefficients of Freon-12 boiling in a vertical tube. Their results showed that pressure and flow rate had very little effect on the coefficients, but as the relative amount of vapor increased from 40 to 100 percent, there was a very significant decrease in the heat transfer coefficients.

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CHAPTER II

EQUIPMENT

The equipment was a flow system in which steam was taken from a main line and passed through a throttle valve, a cooler, the test section, a condenser, and to a weigh tank. The assembly is shown schematically in Figure 1. The steam was supplied from a Babcock and Wilcox FM boiler at a pressure of 200 psig. After passing through a throttle valve the steam was made to flow through a cooler which was intended for use as a method of varying the quality. It was never subsequently used since time limited the investigation to a percent moisture not greater than 6 percent. On either end of the test section an Ellison combination throttling and separating calorimeter was installed for quality determination. Prior to the condenser an orifice of 0.166 inches diameter was installed to maintain the pressure in the test section. A bypass was provided around the orifice to permit adjustment of the flow rate independently of pressure.

The test section as seen in Figures 2 and 4 consisted of a vertical steel tube having an inside diameter of 0.50 inches and an outside diameter of 1.25 inches. Its overall length was approximately $27\frac{1}{2}$ inches, but with an active length of 24 inches for purposes of heat transfer. Recesses were cut in either end of the test section to minimize axial heat loss. The thermocouples in the test section were made of No. 28 gauge chromel-alumel wire, and were installed by the Research

Division of the Babcock and Wilcox Company. Four pairs of thermocouples were installed along the tube, one of each pair at the outer surface of the tube, the other at a known depth below the surface thermocouples. The leads of the thermocouples were threaded through a milled slot and out a $\frac{1}{4}$ " tube welded to the test section. The original thought was that the thermocouples as located would permit determination of the temperature gradient across the wall of the tube, thus the inside surface temperature could be found by extrapolation.

At either end of the test section, thermocouple probes were projected into the fluid stream for measuring the temperature of the steam. These probes were obtained from Leeds and Northrop, and manufactured of iron-constantan wire.

The heating element consisted of four independent heaters, each containing 29.8 feet of No. 18 Nichrome V wire. The wire was wound around the tube in a single layer. Each heater was approximately 6 inches in length. In order to prevent shorting of the heater coils, the wire was insulated with glass sleeving, and a thin layer of mica, 0.0015 inches thick, was wrapped around the tube before the heaters were wound. Power supply to the heaters was controlled by four 115v-15amp Powerstats. See Figure 3. The total power to each heater was measured by portable wattmeters.

The insulation of the test section was accomplished by winding the tube, outside the heaters, with strips of 1/8"

thick asbestos cloth. This was followed by applying a length of standard $1\frac{1}{2}$ " magnesia pipe insulation. The entire section was then covered with an aluminum sheet casing. Provisions were made so that air, heated to approximately the same temperature as the outer surface of the insulation, could be supplied to the space between the insulation and casing, thus minimizing any heat loss due to convection, and reducing the conducted heat loss through the insulation.

CHAPTER III

OPERATING PROCEDURE

There were four primary variables associated with the apparatus: heat flux, flow rate, quality, and pressure. It was found, however, during preliminary experimentation, that the minimum moisture content of the steam available for use was approximately 6%. Interest at this time was primarily centered on the region between 0 and 10 percent moisture, thus necessitating throttling the steam to lower pressures in order to obtain a given quality at the entrance of the test section. Such an arrangement meant that only the heat flux, pressure, and flow rate could be varied independently of one another. The only other limiting factor associated with the equipment was, that due to the design of the heaters and the power supply used, the maximum heat input was restricted to 1 kw per heater, or a total of 4 kw.

The procedure was to select an approximate quality at the entrance of the test section by setting the throttle valve accordingly. The flow rate was either taken as that provided by the pressure across the orifice in the line to the condenser or adjusted by means of the by pass around the orifice. Power was then supplied to the heaters and the system was allowed to come to a steady-state condition.

The approach to steady-state condition was determined by noting the change in the millivolt reading of the thermocouples contained in the test section. When they became

THE HISTORY OF THE CITY OF BOSTON

The history of the city of Boston is a subject of great interest and importance. It is a city of many centuries, and its history is full of interesting events. The city was founded in 1630, and has since that time been a center of commerce and industry. It has been the seat of many important events, and has played a large part in the history of the United States. The city is now one of the largest and most important cities in the world, and its history is a subject of great interest to all who are interested in the history of the United States.

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approximately constant, steady-state was considered to exist. It normally took from 1/2 to 3/4 hours before the system reached equilibrium. It was never possible to keep temperatures constant over a period longer than a few minutes, even though heat input and flow rate appeared unchanged. Each run was made at constant flow rate, heat flux, pressure and quality of the inlet steam.

For each run, temperatures, pressures, flow rate, and power input were recorded. Data points were taken at various settings of power inputs, flow rates, and pressures. Most of the runs were concluded at a power input such that the minimum change in quality which occurred through the test section was 3%.

CHAPTER IV

METHOD OF CALCULATION

The fundamental law of heat transfer by convection may be stated by the mathematical expression:(4)

$$q = hA \Delta T \quad (2)$$

The factor of proportionality h , defined in the equation, is called "the film coefficient of heat transfer." It is this coefficient which was computed.

For each set of conditions imposed on the test section, the density of heat flux, the inside surface temperature of the tube, the ΔT , the quality of the steam, and the local coefficient of heat transfer were calculated. Each of these factors was evaluated at four positions corresponding to the locations of the thermocouples inside the tube wall.

The power dissipated in the heating elements was obtained from wattmeter measurements. The heat flux density was then calculated from the power dissipated and the dimensions of the heat transfer surface.

Early in the process of collecting data for the investigation, it was discovered that the thermocouple pairs in the test section were indicating unreasonable temperature gradients through the tube wall. It became necessary to calculate the inner surface temperature by applying to the depth thermocouple a computed temperature drop corresponding to the heat input. The temperature drop was calculated using

the Fourier conduction equation for steady-state heat transmission: (4)

$$q = -k A \frac{dt}{dx} \quad (3)$$

For a circular tube, this equation may be integrated to give:

$$q = \frac{2\pi k L \Delta T}{\ln r_o/r_i} \quad (4)$$

The depth thermocouples were shielded from any direct influence of the heaters, and thus were considered to be more accurate than those on the surface of the tube.

The temperature of the steam was measured at both the entrance and exit of the test section, however, these measurements were consistently erratic and failed to give desired results. It was felt that greater reliability would result from assuming the saturation temperature corresponding to the absolute pressure in the test section for computing the ΔT .

In the preliminary phase of the investigation it was discovered that the reproducibility of the calorimeters, for determination of quality below the level which could be found by the throttling principle alone, was rather poor. With results taken over a long period of time it was found that the average quality of the steam at the test section, at a pressure of 200 psig, was 94%. Further use of the calorimeters was discontinued, and this value of 94% for quality was used as a basis for all further computations. The quality of the steam, at each point in the test section corresponding to the location of the thermocouples, was computed by adjusting the

enthalpy of the incoming steam in an amount equal to the heat input to the test section. The quality at pressures lower than 200 psig was determined by the use of the Mollier Diagram.

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CHAPTER V

CONCLUSIONS AND RESULTS

One hundred runs were made of which forty-six were successful. The remainder of the runs were eliminated because the tube wall temperatures obtained indicated that steady-state conditions had not yet been reached. The ranges of pressure, temperature, quality, and mass rate of flow are given in Table I of Appendix I.

In the process of collecting data a number of experimental errors became apparent. These errors, however, did not exceed the range of expectancy ordinarily associated with this type of investigation. The temperature determinations were expected to be in error at least 2 to 3%. This was not serious as long as the temperature difference between the fluid and the wall surface was large. At small temperature differences care had to be taken to insure that unreasonable results were not obtained. Another error is introduced in the calculation of the transfer coefficient by neglecting axial heat loss from the test section. In estimating this loss it was found to be small, in the range of 10 to 40 BTU/hr. The error due to the radial heat loss was approximately 2% of the input power.

Curves for the heat transfer coefficient versus the percent moisture were made for each mass velocity. In addition a plot was made of the transfer coefficient versus the temperature difference between the wall and the fluid, and one

THE
HISTORY OF THE
CITY OF BOSTON

From the first settlement of the city in 1630 to the present time, the history of Boston is a story of growth and development. The city has been a center of commerce, industry, and culture, and has played a significant role in the history of the United States. The city's location on a peninsula in the harbor of Massachusetts Bay made it a natural center of trade and commerce. The city's growth was rapid, and by the mid-17th century, it was one of the largest and most important cities in the colonies. The city's history is marked by many significant events, including the Boston Tea Party, the Battle of Bunker Hill, and the American Revolution. The city's culture is rich and diverse, and it has been a center of education, science, and the arts. The city's history is a testament to the resilience and spirit of its people, and it continues to shape the city's identity and future.

of the transfer coefficient versus the mass rate of flow. These curves are seen in Figures 6 through 12. The physical properties of steam-water mixtures were not available, thus preventing any correlation of the transfer coefficient with any existing data.

The curves depicting the heat transfer coefficient versus the percent moisture show a very gradual rise in the coefficient up to moisture content between one and two percent. Above this value the coefficient increased rapidly as the moisture increased from two to five percent. A possible explanation of this phenomenon is that at high qualities there are only small droplets of water in the fluid, but not enough to materially disrupt the boundary layer between the steam and the wall of the test section. As the amount of moisture in the steam increases, the water droplets become larger and begin to rupture the boundary layer and collect on the wall of the test section, with a consequent increase in the heat transfer coefficient. The sharp change in slope of the curve is the point at which the water begins to collect in droplets on the tube wall. With a further increase in moisture, there is an approach to annular flow where the boundary layer becomes completely liquid. At this point the transfer coefficient should reach a maximum and then start to decrease. Fully developed annular flow was never reached in this experiment.

Figure 11 shows the influence of mass flow rate on the heat transfer coefficient. It is surprising to note that the coefficient decreased as the flow rate increased to a value of about 200 lbs/hr. With flow rates above this value, the coefficient increased. Only one point on the curve was established beyond this transition region. However, the predicted curve is extrapolated (dotted line) on the plot. The initial decrease in the coefficient was possibly due to the water droplets on the wall being flattened out as the flow rate increased and thereby forming a liquid boundary layer. As the mass flow rate increased beyond 200 lbs/hr the velocity of the gas was apparently sufficient to start wiping the water droplets off the wall instead of just flattening them out. This is how the continued upward trend in the curve was predicted.

Figure 12 indicates curves of the transfer coefficient versus the difference in temperature between the tube wall and the fluid. This shows the correlation between the heat transfer coefficient and temperature difference, for a given range in quality of about five percent. These curves appear to be in the region of unstable film boiling when compared to the boiling curve for free convection⁽⁷⁾, but the coefficients determined in the investigation are necessarily higher than those of the boiling curve due to the influence of forced convection.

The available literature and results of this investigation permits the conclusion that a maximum heat transfer coefficient exists with respect to the moisture content of the mixture. This point has not been within the range of the subject investigation. The location of this maximum represents an optimum for future design of certain types of heat transfer equipment.

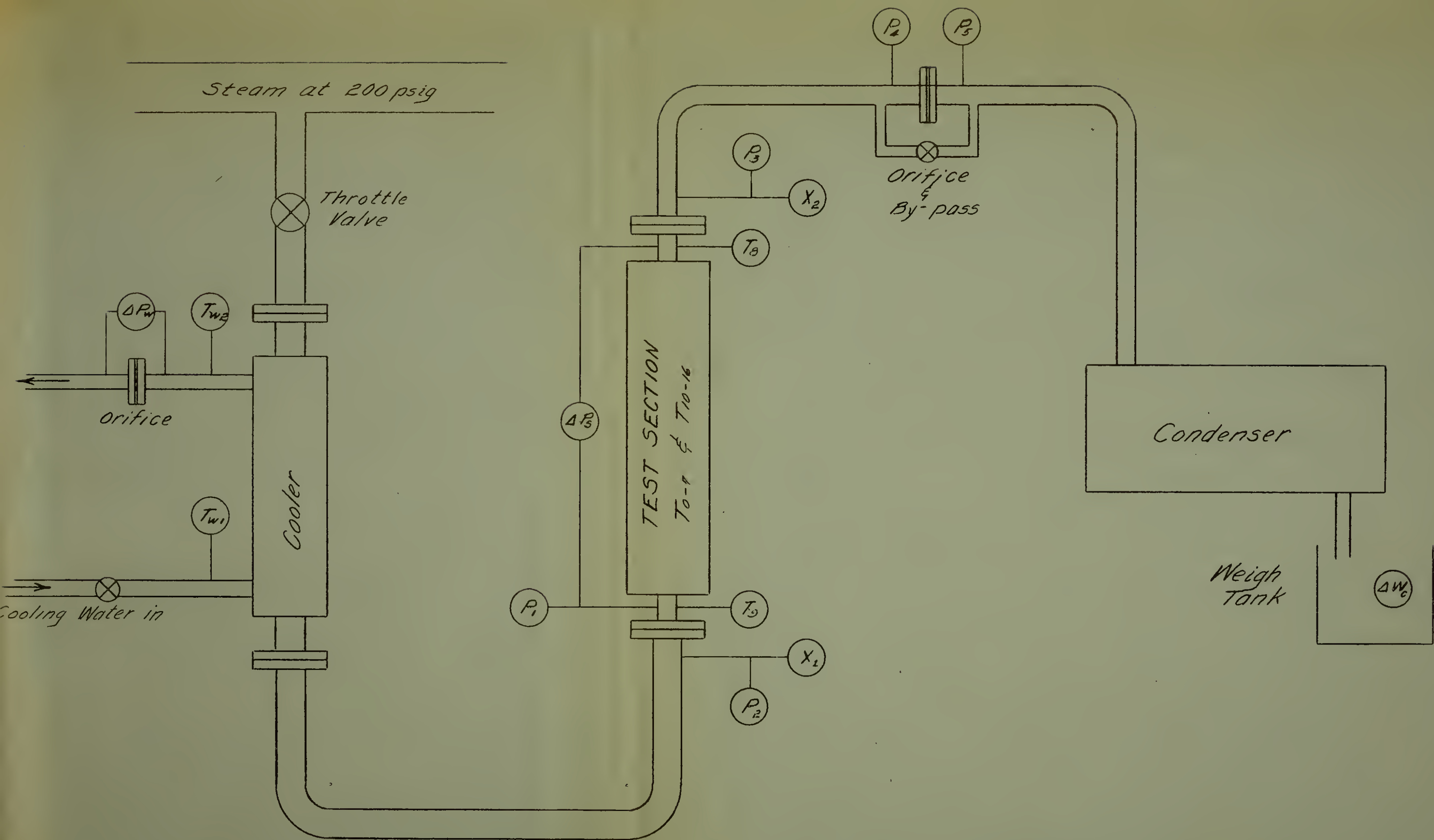
Additional research would be amply justified in order that the peak in this curve be definitely established.

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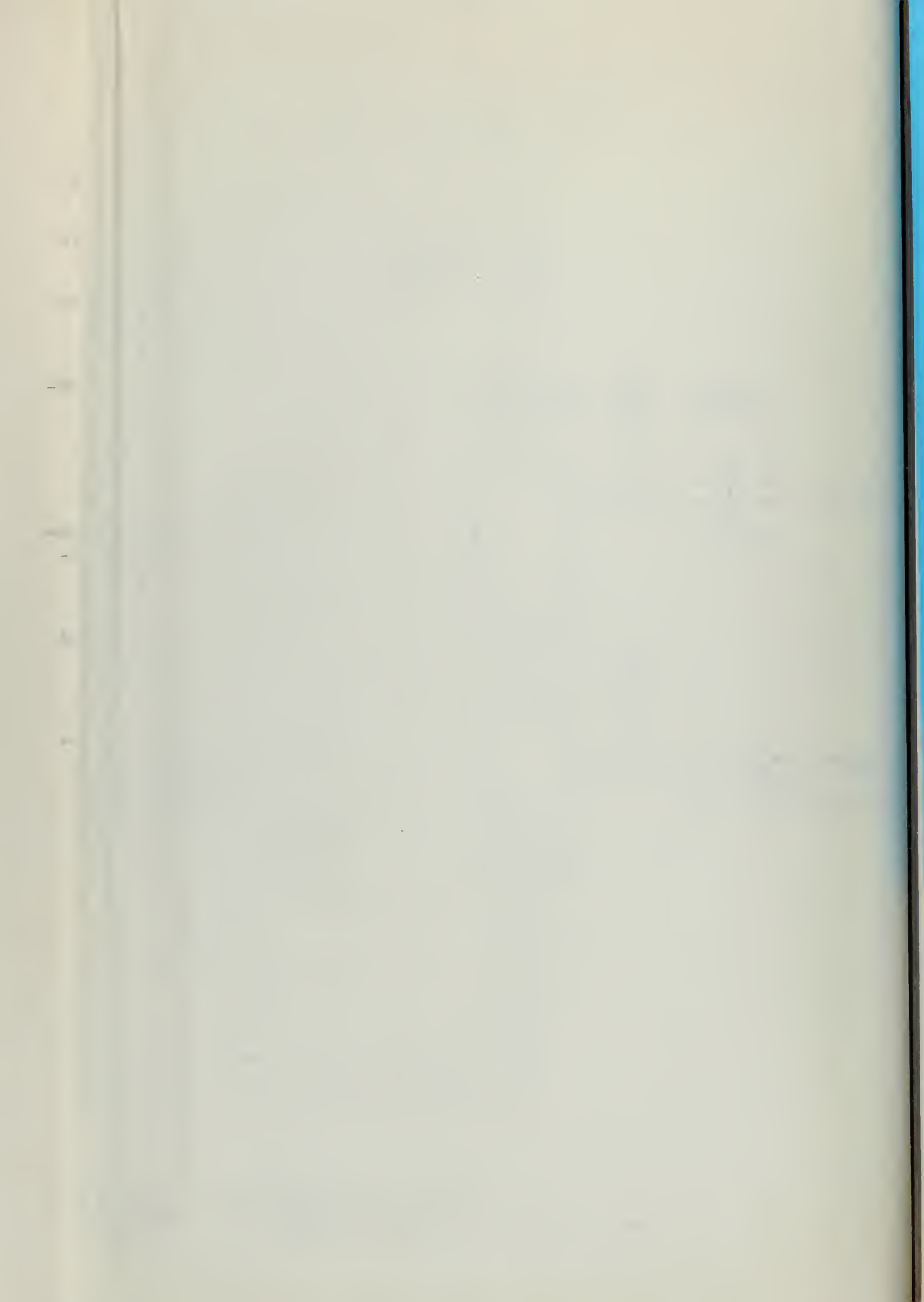
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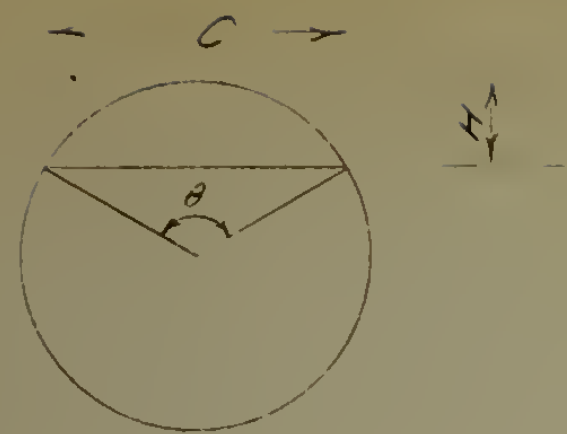
CHAPTER 1

1. The first part of the book is devoted to a general introduction to the subject of the book. It is divided into two main parts: the first part deals with the general principles of the subject, and the second part deals with the specific details of the subject.
2. The second part of the book is devoted to a detailed discussion of the various aspects of the subject. It is divided into three main parts: the first part deals with the general principles of the subject, the second part deals with the specific details of the subject, and the third part deals with the application of the subject to various fields of study.
3. The third part of the book is devoted to a detailed discussion of the various aspects of the subject. It is divided into three main parts: the first part deals with the general principles of the subject, the second part deals with the specific details of the subject, and the third part deals with the application of the subject to various fields of study.
4. The fourth part of the book is devoted to a detailed discussion of the various aspects of the subject. It is divided into three main parts: the first part deals with the general principles of the subject, the second part deals with the specific details of the subject, and the third part deals with the application of the subject to various fields of study.
5. The fifth part of the book is devoted to a detailed discussion of the various aspects of the subject. It is divided into three main parts: the first part deals with the general principles of the subject, the second part deals with the specific details of the subject, and the third part deals with the application of the subject to various fields of study.
6. The sixth part of the book is devoted to a detailed discussion of the various aspects of the subject. It is divided into three main parts: the first part deals with the general principles of the subject, the second part deals with the specific details of the subject, and the third part deals with the application of the subject to various fields of study.
7. The seventh part of the book is devoted to a detailed discussion of the various aspects of the subject. It is divided into three main parts: the first part deals with the general principles of the subject, the second part deals with the specific details of the subject, and the third part deals with the application of the subject to various fields of study.
8. The eighth part of the book is devoted to a detailed discussion of the various aspects of the subject. It is divided into three main parts: the first part deals with the general principles of the subject, the second part deals with the specific details of the subject, and the third part deals with the application of the subject to various fields of study.
9. The ninth part of the book is devoted to a detailed discussion of the various aspects of the subject. It is divided into three main parts: the first part deals with the general principles of the subject, the second part deals with the specific details of the subject, and the third part deals with the application of the subject to various fields of study.
10. The tenth part of the book is devoted to a detailed discussion of the various aspects of the subject. It is divided into three main parts: the first part deals with the general principles of the subject, the second part deals with the specific details of the subject, and the third part deals with the application of the subject to various fields of study.



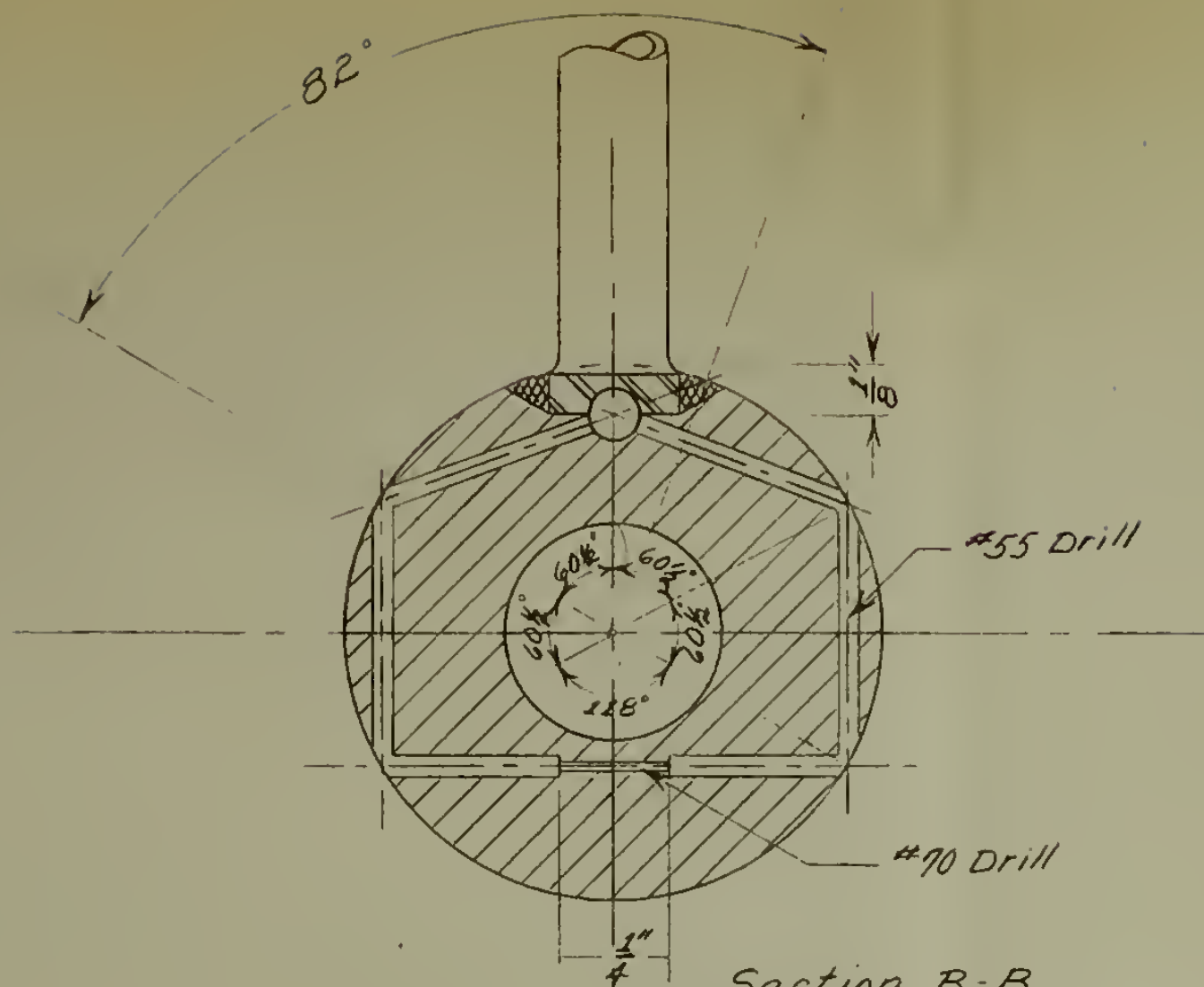
SCHEMATIC - HEAT FLOW APPARATUS - FIGURE 1





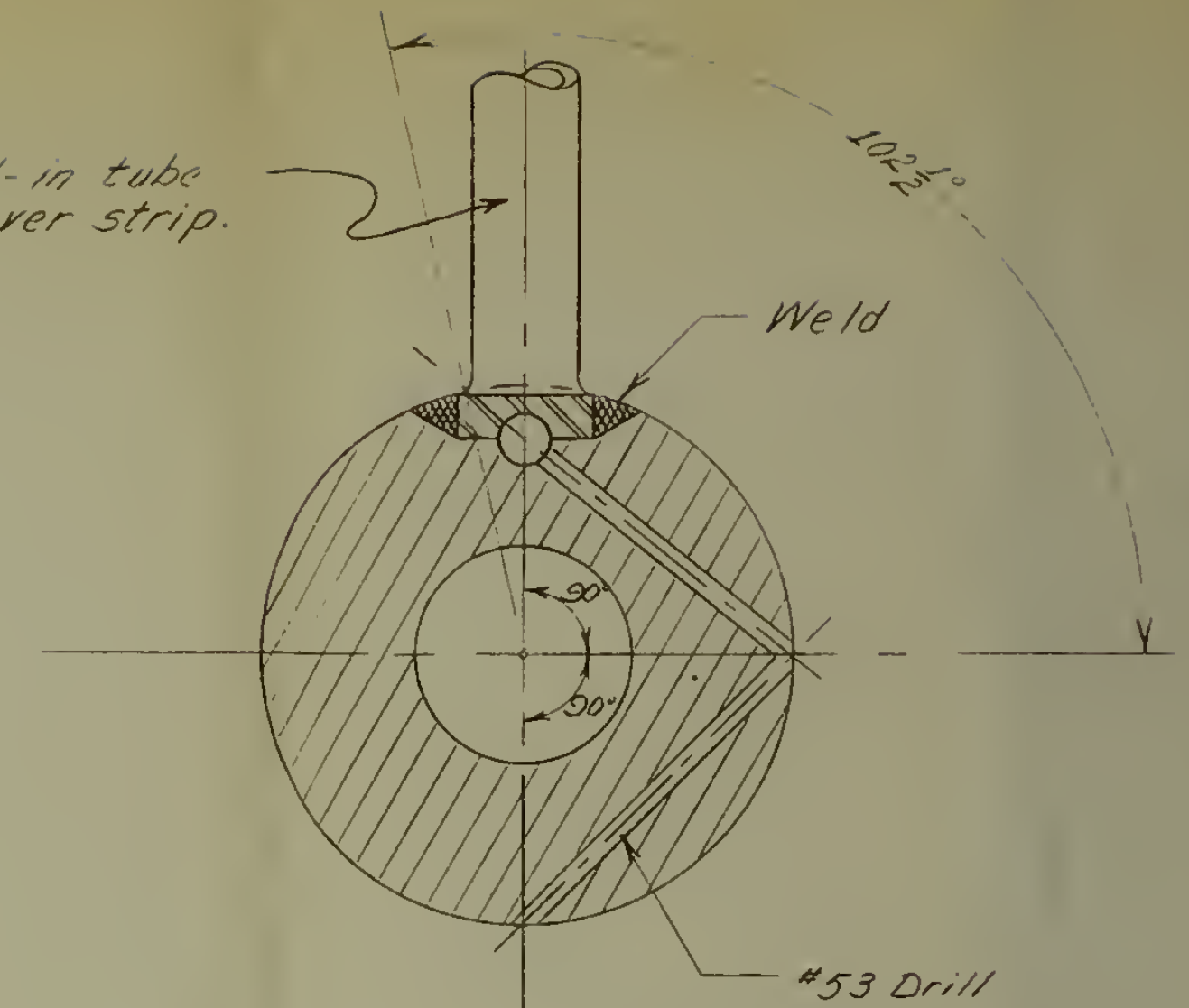
Angle θ	C	H
$60\frac{1}{2}^\circ$	0.6298"	0.0851"
90°	0.8838"	0.1831"
118°	1.071"	0.3031"

Chord Table

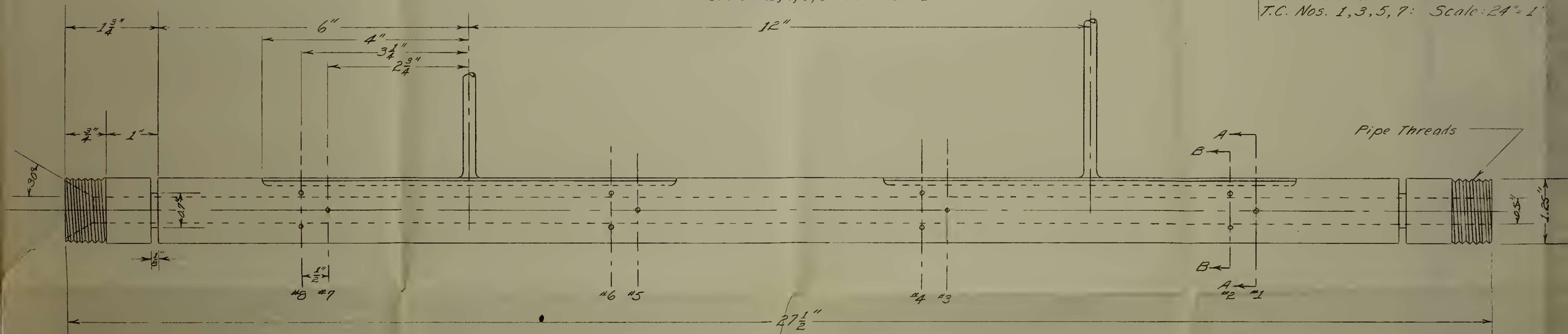


Section B-B
T.C. Nos. 2, 4, 6, 8 : Scale: 24" = 1'

1/4" O.D. Lead-in tube
welded to cover strip.

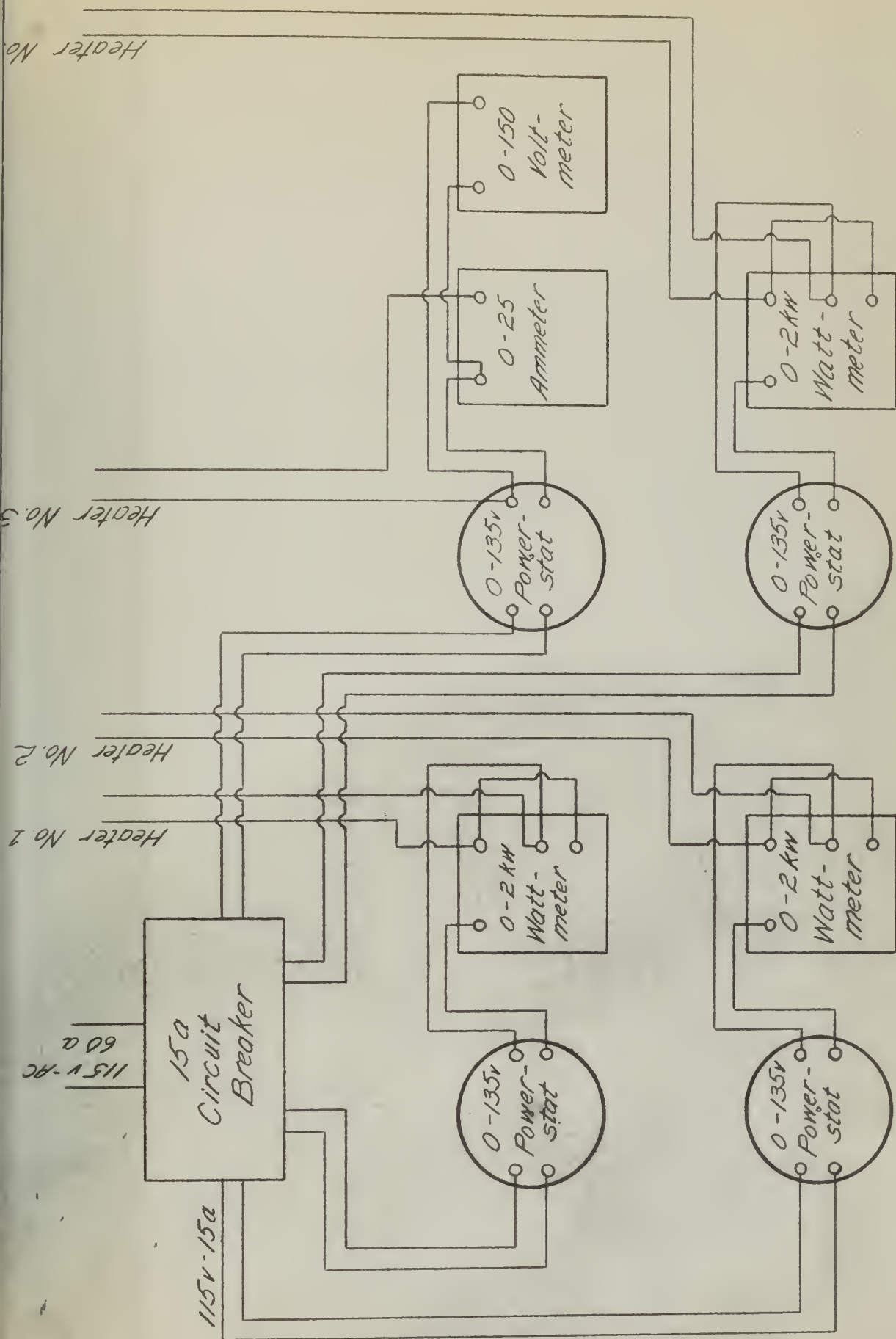


Section A-A
T.C. Nos. 1, 3, 5, 7 : Scale: 24" = 1'

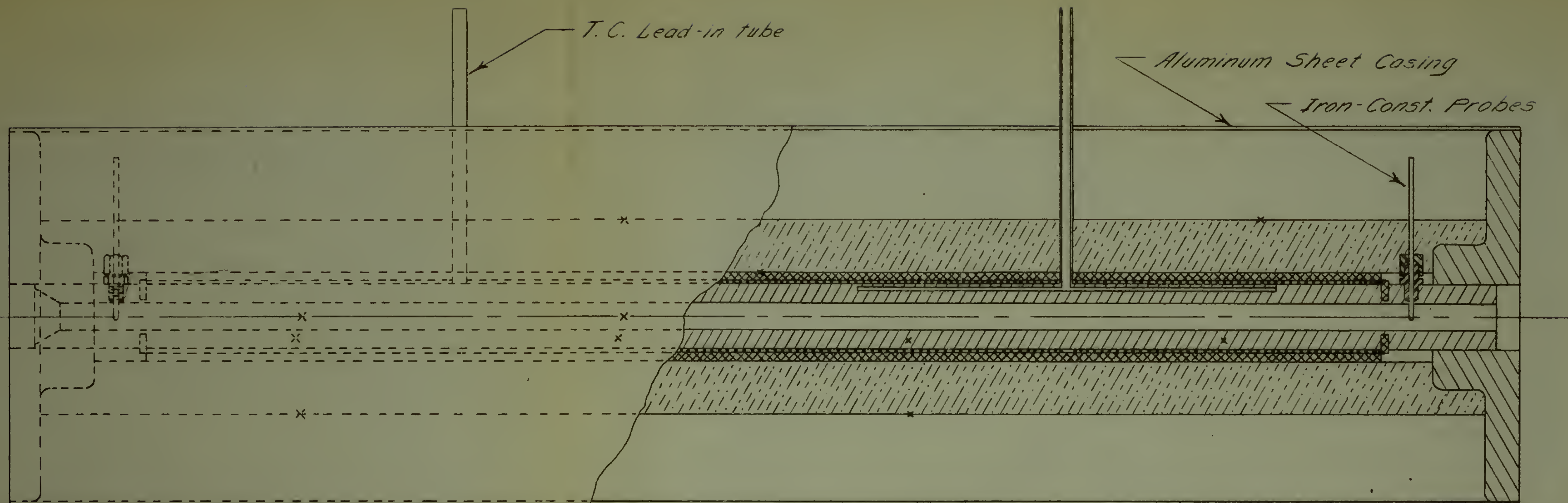



Scale: 9" = 1'


DETAIL of TEST SECTION - FIGURE 2




SCHEMATIC of HEATER CONTROL CIRCUIT
FIGURE 3



 Brass Fittings

 Steel tube & flanges

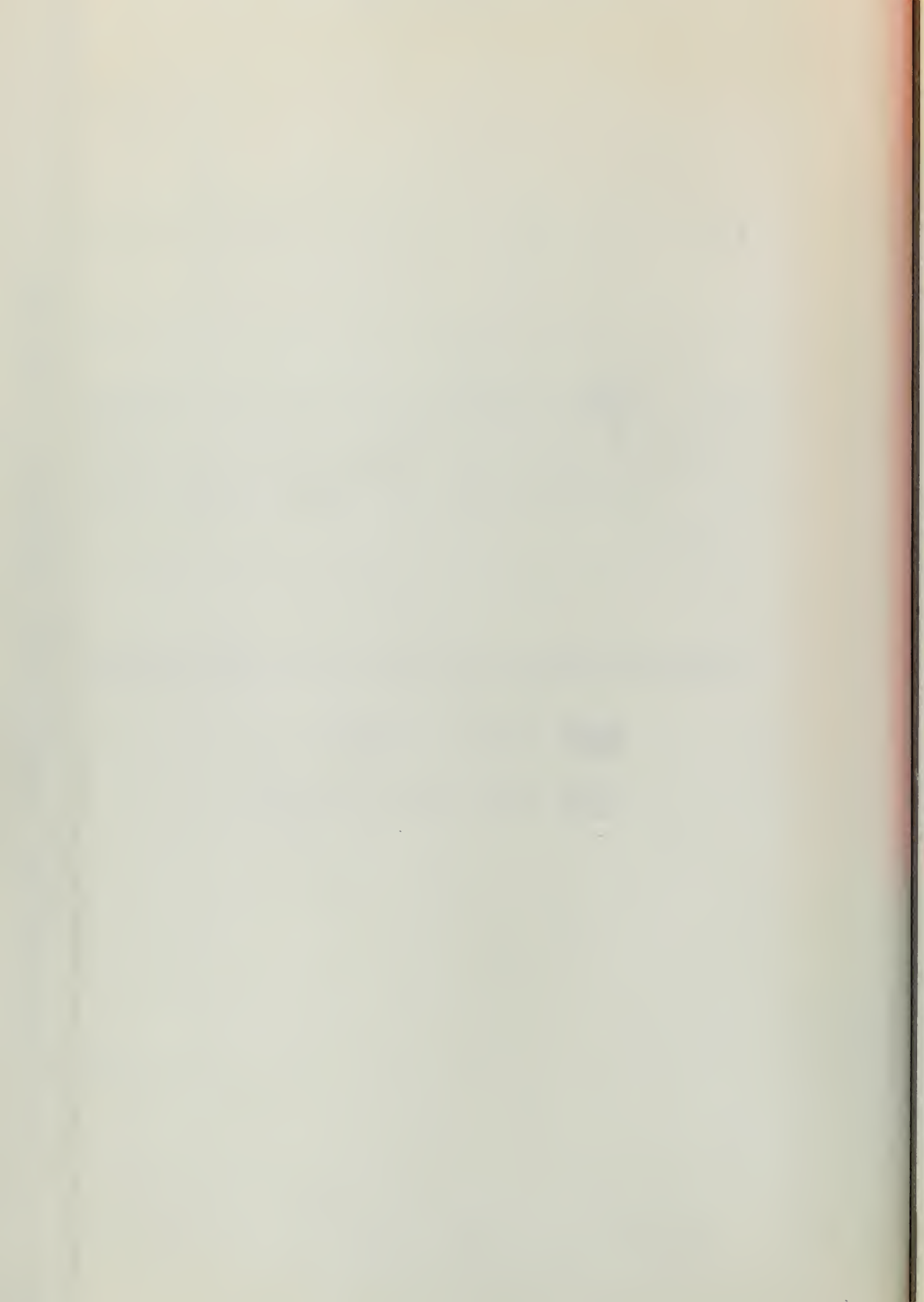
 Asbestos

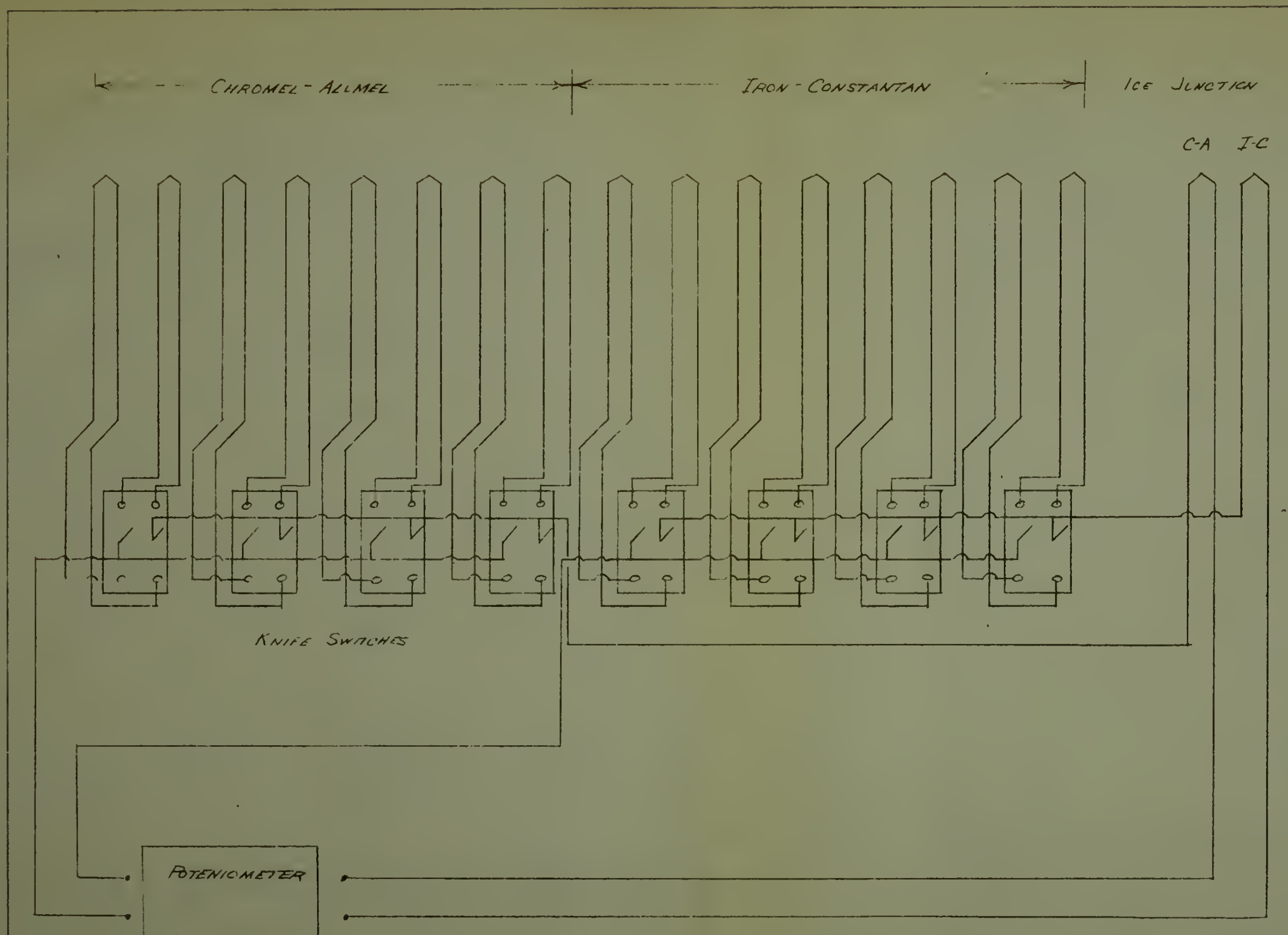
 85% Magnesia

x x Thermocouples

..... Heater Wires

TEST SECTION - FIGURE 4





SCHEMATIC - THERMOCOUPLE CIRCUIT - FIGURE 5

h - HEAT TRANSFER COEFFICIENT IN BTU/HR-FT²-°F

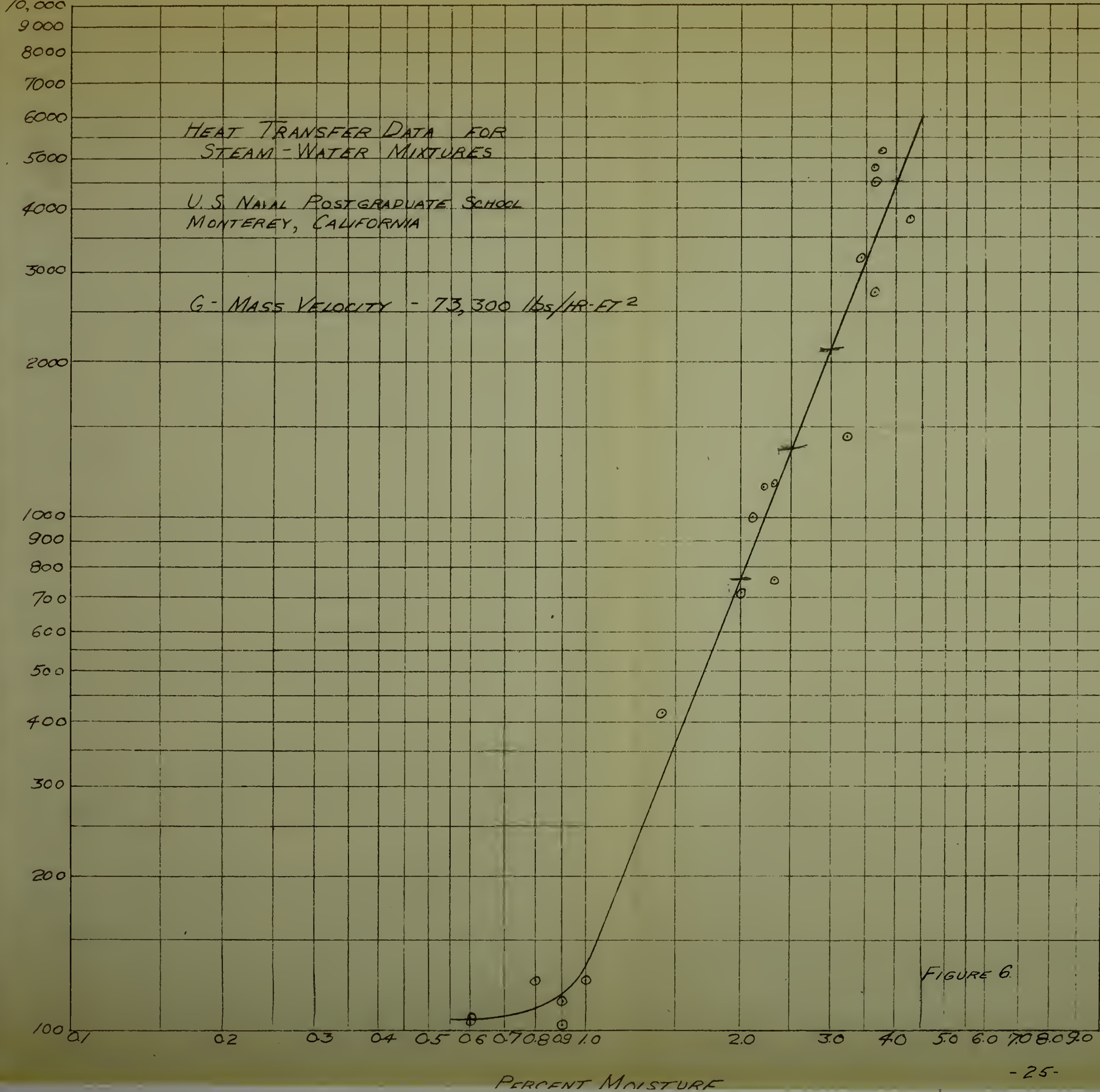
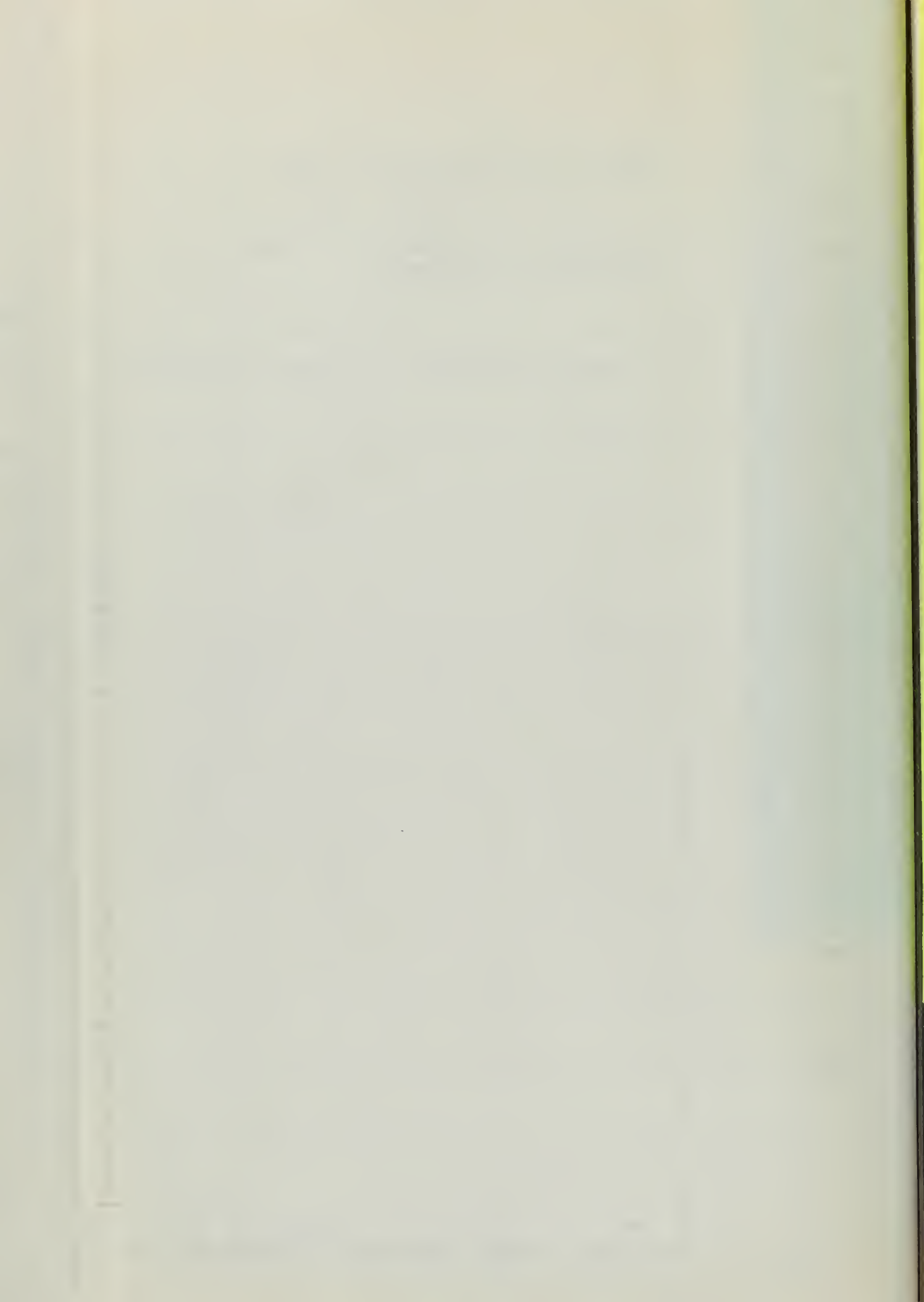


FIGURE 6



h - HEAT TRANSFER COEFFICIENT IN BTU/HR-FT²-°F

HEAT TRANSFER DATA FOR
STEAM-WATER MIXTURES

U.S. NAVAL POSTGRADUATE SCHOOL
MONTEREY, CALIFORNIA

G - MASS VELOCITY - 87,900 lbs/HR FT²

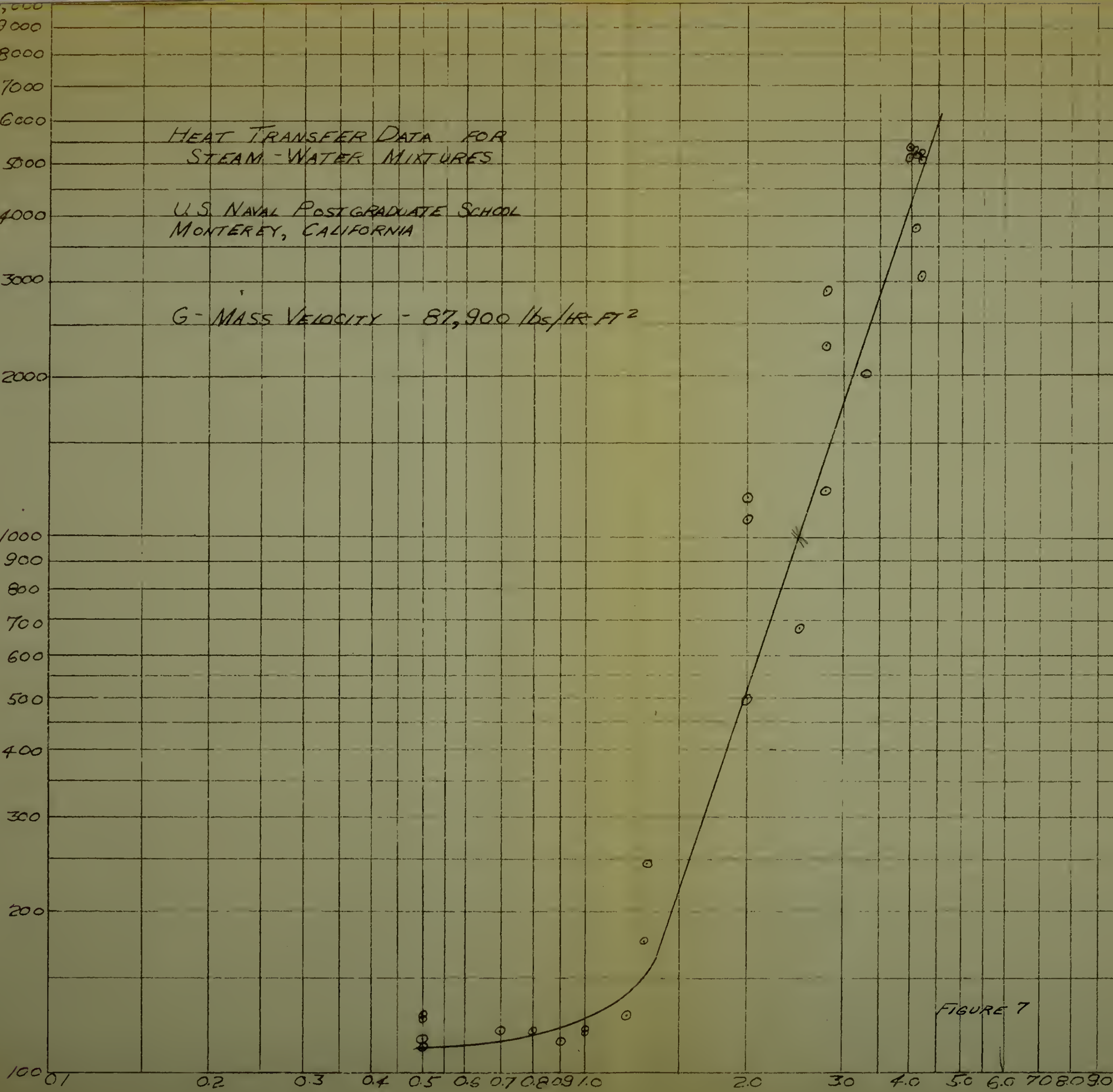
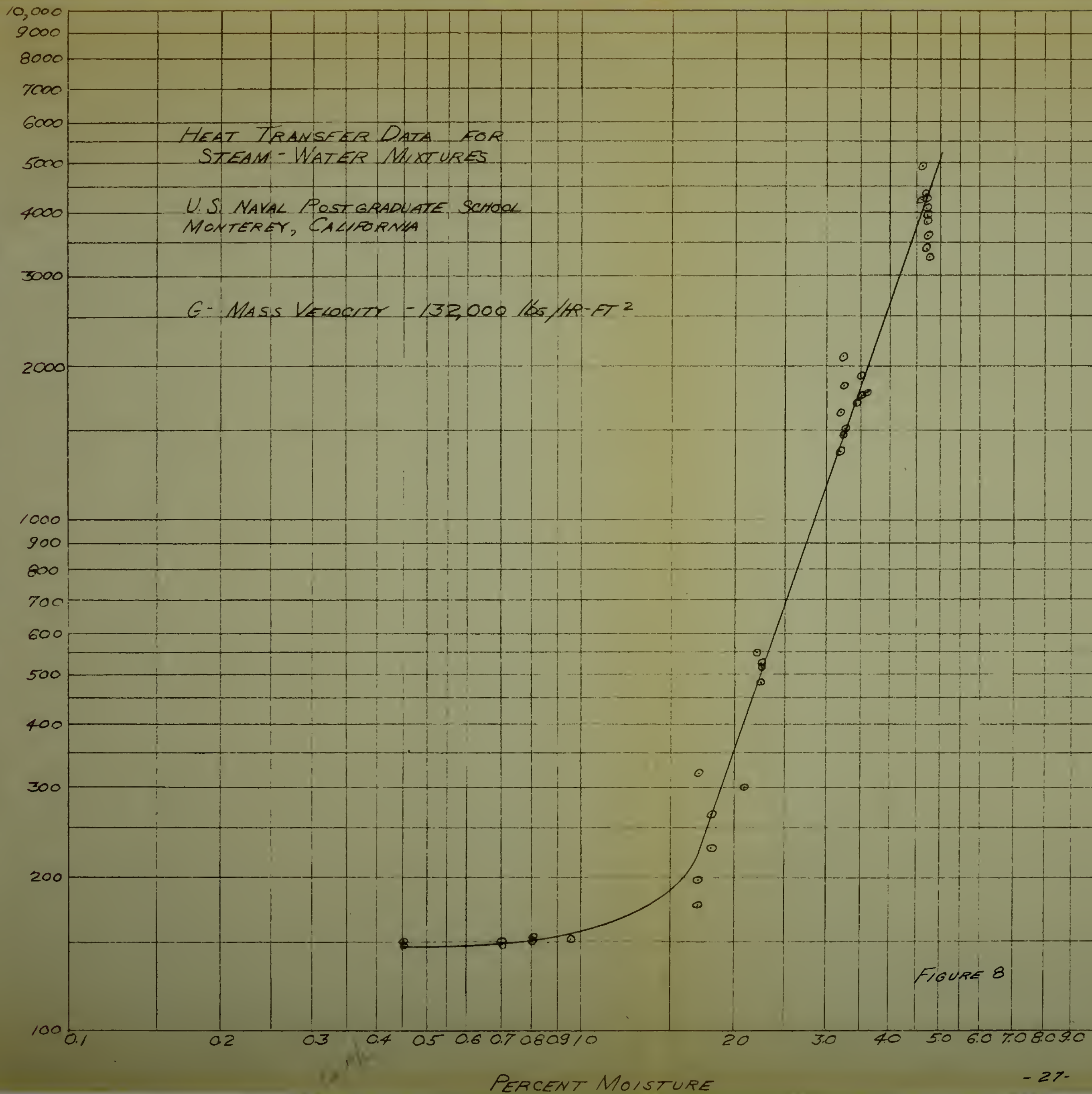


FIGURE 7

PERCENT MOISTURE

h - HEAT TRANSFER COEFFICIENT IN BTU/HR-FT²-°F



h_1 - HEAT TRANSFER COEFFICIENT IN BTU/HR-FT²-°F

HEAT TRANSFER DATA FOR
STEAM-WATER MIXTURES

U.S. NAVAL POSTGRADUATE SCHOOL
MONTEREY, CALIFORNIA.

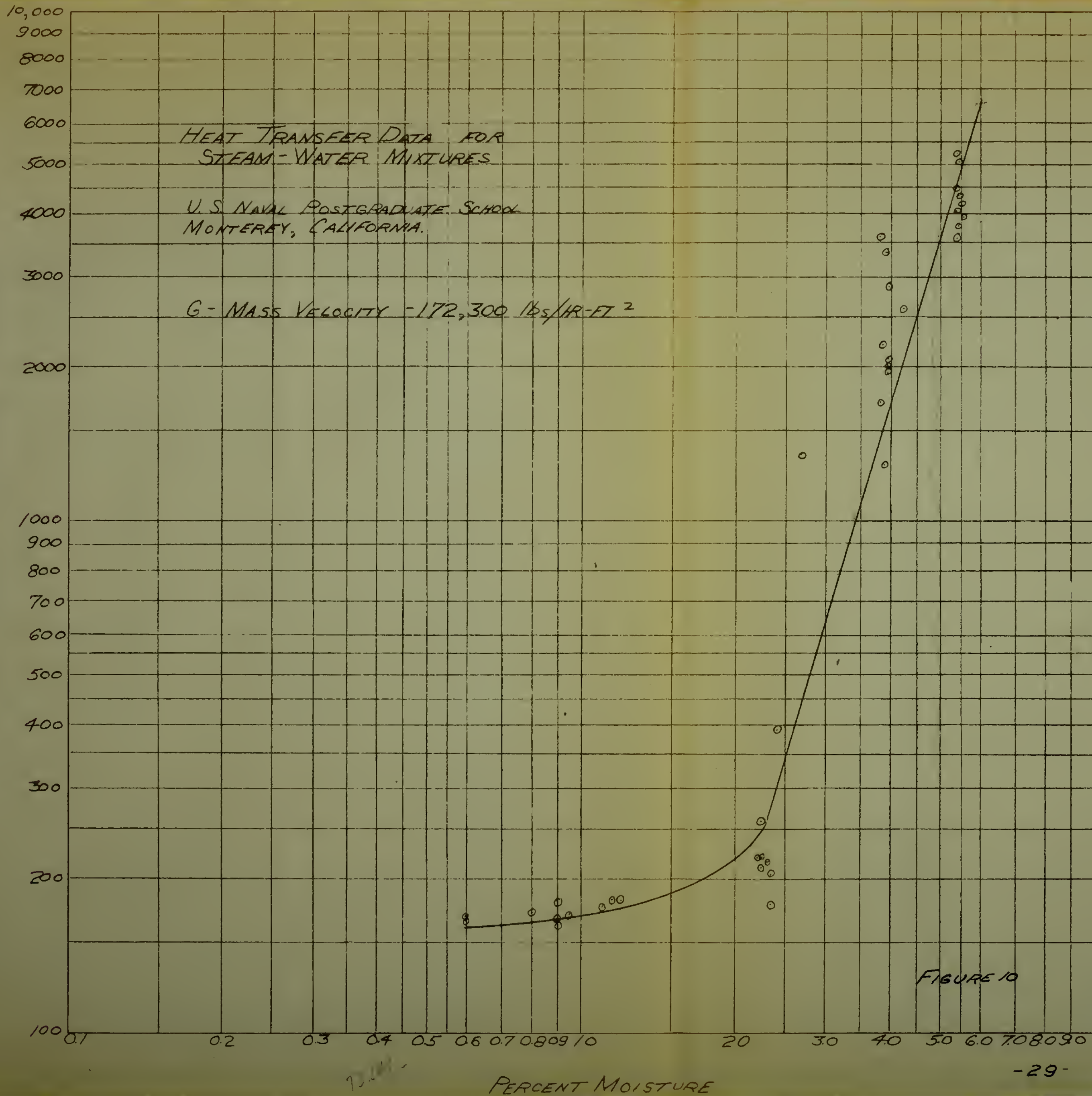
G - MASS VELOCITY - 146,500 LBS/HR-FT²

100 0.1 0.2 0.3 0.4 0.5 0.6 0.7 0.8 0.9 1.0 2.0 3.0 4.0 5.0 6.0 7.0 8.0 9.0

PERCENT MOISTURE

FIGURE 9

h - HEAT TRANSFER COEFFICIENT IN BTU/HR-FT²-°F



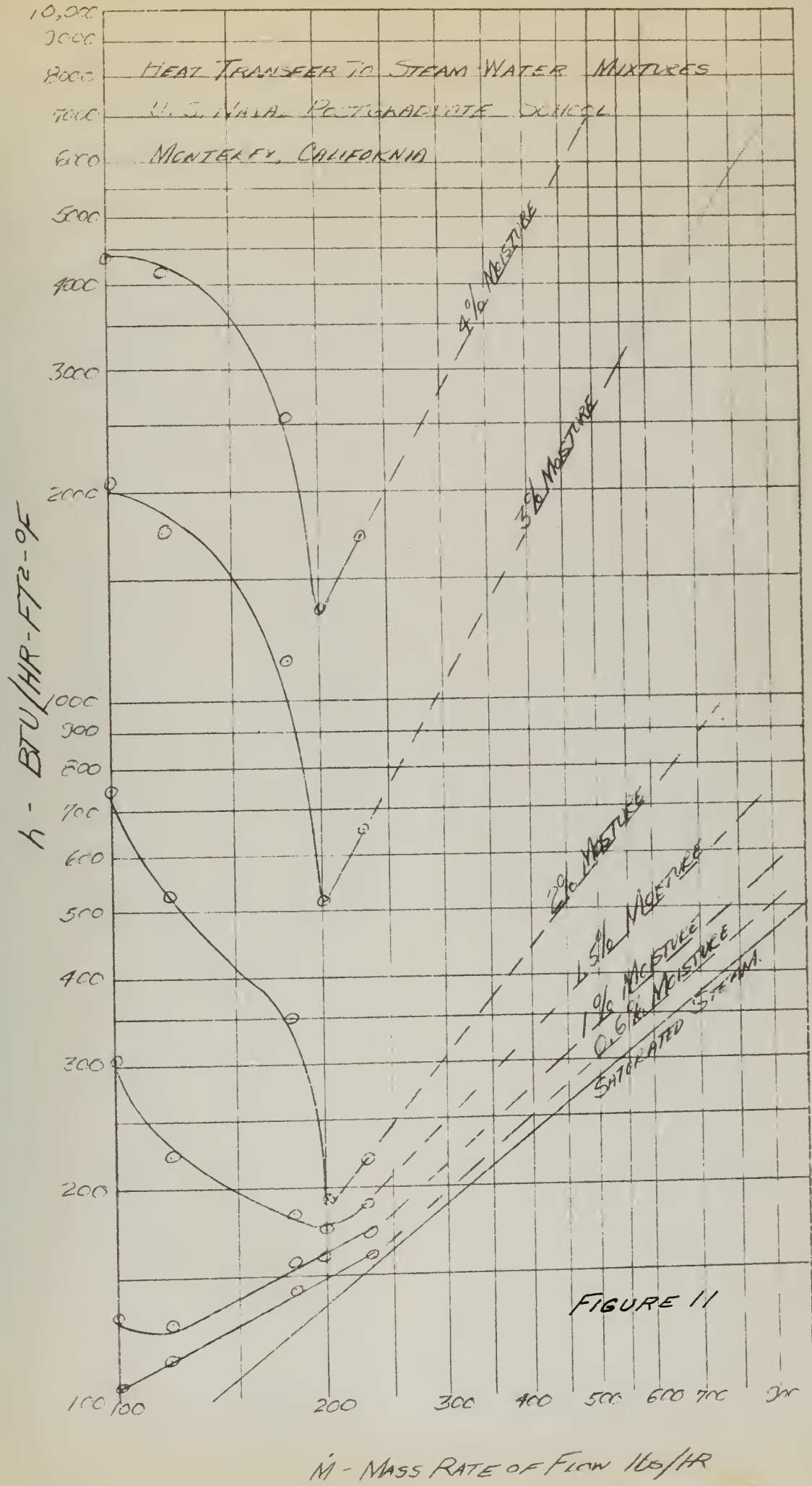
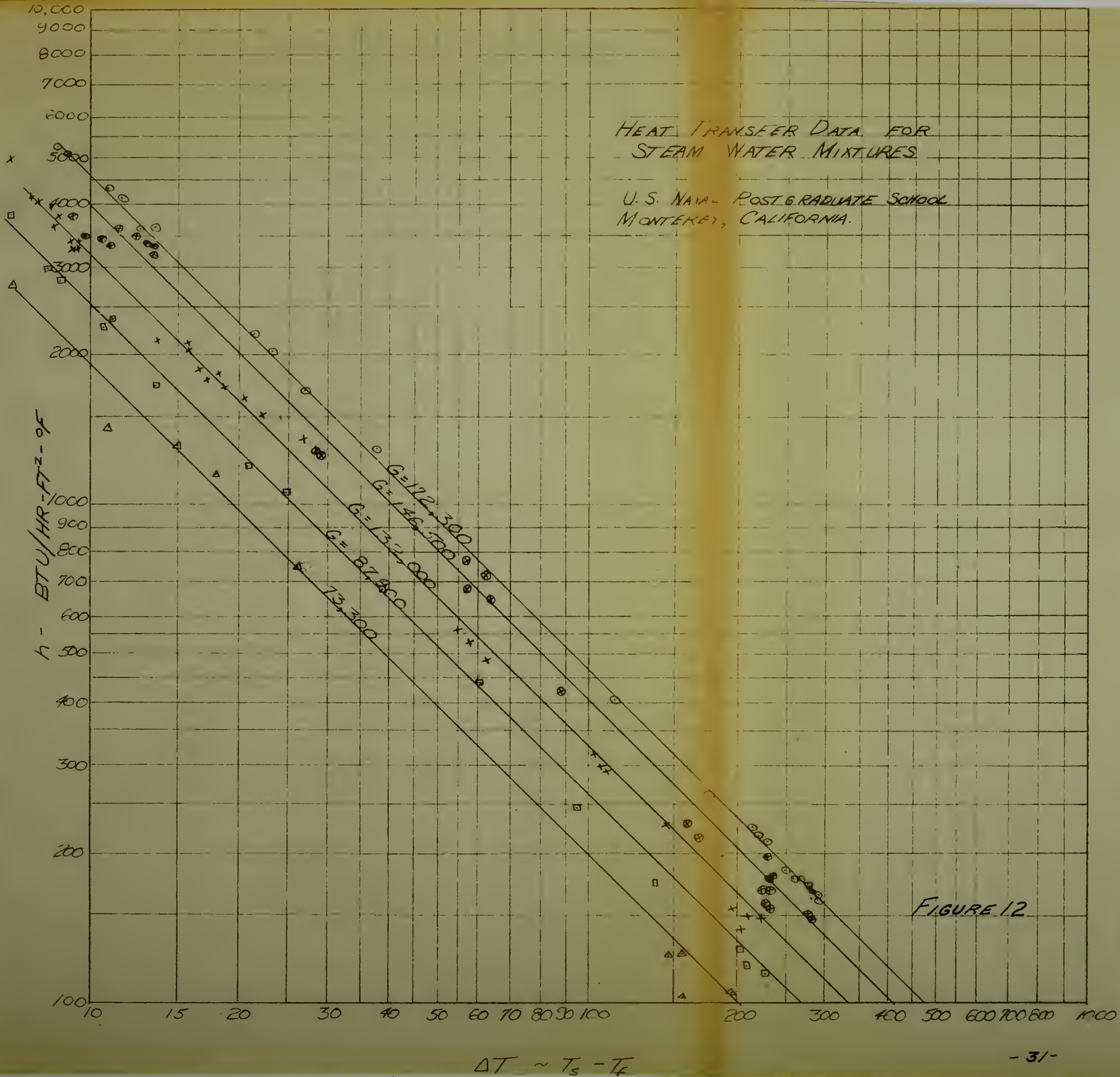


FIGURE 11





APPENDIX

TABLE I

SUMMARIZED DATA

Mass Flow Rate 100 lbs/hr.

Run No.	Press psia	Inlet Temp of	Outlet Temp of	Test Sta.	Test Sec. No.	Test Temp of	Ts	x	h	Q/A
1	103	329.96	377	1		355.7	338.2	.966	3170	26,050
						342.9				
				2		456.1				
						397.4	392.7	.986	415	25,900
				3		633.2	598	Supht	--	25,800
						606.4				
				4		662.9	652.5	Supht	--	26,050
						655.6				
2	97	325.6	351	1		345.0	329.6	.963	5200	20,800
						333.3				
				2		363.5	343.6	.977	1160	20,800
						347.3				
				3		551.5	519.8	.994	107	20,800
						523.5				
				4		577.3	573	Supht	--	20,800
						574.0				
3	98	326.35	351	1		341.5	330.98	.969	4500	20,800
						333.3				
				2		352.5	347.15	.979	1000	20,800
						348.8				
				3		512.5	507	.991	115	20,800
						509.0				
				4		568.5	567	Supht	--	20,800
						567.5				

0 1 2 3 4 5 6 7 8 9

1 2 3 4 5 6 7 8 9

1 2 3 4 5 6 7 8 9

1 2 3 4 5 6 7 8 9

1 2 3 4 5 6 7 8 9

1 2 3 4 5 6 7 8 9

1 2 3 4 5 6 7 8 9

1 2 3 4 5 6 7 8 9

1 2 3 4 5 6 7 8 9

1 2 3 4 5 6 7 8 9

1 2 3 4 5 6 7 8 9

1 2 3 4 5 6 7 8 9

1 2 3 4 5 6 7 8 9

Run No.	Press psia	Inlet Temp °F	Outlet Temp °F	Test Sta. No.	Sec. Temp °F	T _s	x	h	Q/A
4	98	326.35	345	1	341.0	330.5	.964	4840	19,800
				2	334.0				
					355.5				
				3	345.0	341.5	.978	1310	19,800
					510.3				
				4	485.0	481.3	.992	127.9	19,800
					541.0				
5	98	326.35	340	1	538.2	537	Supht	--	18,150
					348.5				
				2	336.5	333.2	.964	2740	18,150
					372.8				
				3	355.3	352.5	.977	750	18,150
					496.5				
				4	475.5	472.2	.990	125.1	18,150
					541.0				
6	98	325.6	325.6	1	532.5	530	Supht	--	18,150
					339.5				
				2	332.5	329.7	.958	3820	16,200
					345.5				
				3	339.3	336.5	.968	1431	16,200
					368.0				
				4	351.5	348.6	.980	714	16,200
					488.3				
					483.8	480.9	.941	103.4	16,200
Mass Flow Rate = 120 lbs/hr.									
7	115	338.1	338.1	1	359.2	342.7	.958	5210	20,800
				2	346.4				
					368.0				
				3	354.5	348.47	.957	2000	20,800
					400.0				
				4	385.0	379	.980	500	20,800
					488.3				
					483.8	480.9	.941	103.4	16,200

Run No.	Press psia	Inlet Temp OF	Outlet Temp OF	Test Sta.	Sec No.	Test Temp OF	Sec. OF	Ts	x	h	O/A
8	125	344.3	367		1	372.9		351.2	.959	3800	26,050
					2	355.9					
					3	404.6		382.7	.925	679	26,200
					4	387.4		569.2	.991	116	26,050
9	128	346.1	363			602.4					
						573.4					
						629.6					
						623.6		618.9	Supht	--	26,050
					1	373.2		354.7	.958	3040	25,800
					2	359.4					
					3	390.6		366.7	.972	1210	25,800
					4	371.4		548.2	.988	129.1	25,800
10	123	343.1	359			580.2					
						552.9					
						512.1		601.9	Supht	--	26,050
						606.6					
					1	367.2		347.6	.959	5250	23,600
					2	351.9					
					3	373.3		353.6	.972	2290	23,650
					4	357.9		477.7	.987	176.5	23,600
11	125	344.3	359			519.4					
						481.9					
						577.6		567.9	Supht	--	23,500
						569.9					
					1	365.7		349.1	.958	5070	23,600
					2	353.4					
					3	372.1		352.6	.972	2860	23,500
					4	356.9					
						488.9					
						443.2		439	.987	247	23,500
						572.9					
						564.6		562	Supht	--	23,500

Run No.	Press psia	Inlet Temp OF	Outlet Temp OF	Test Sta. No.	Test Sec Temp OF	T _s	x	h	Q / A
12	126	344.9	345.5	1	372.9	350.3	.960	5140	32,000
				2	355.9				
					400.1				
				3	379.9	374.3	.980	1095	32,000
					629.9				
				4	598.9	593.3	.995	127.5	32,000
					663.9				
					657.3	655	Supht	--	32,000
13	126	344.9	344.9	1	379.1	350.8	.960	5300	31,900
				2	376.5				
					406.1				
				3	379.1	373.5	.980	1190	31,300
					631.9				
					595.6	590	.995	126.9	31,300
				4	667.6				
					661.1	660	Supht	--	31,300

Mass Flow Rate = 180 lbs/hr

14	192	378.4	395	1	417.4	388	.953	4340	42,700
				2	394.4				
					518.1	412	.970	1240	41,600
				3	418.4				
					667.4	626	.987	169	41,600
				4	635.9				
					685.4	680	Supht	--	41,600
					681.4				
15	187	376.2	381	1	401.54	384.54	.953	4000	33,400
				2	384.54				
					412.34				
				3	395.34	398.33	.967	1505	33,300
					545.04	499.84	.982	270	33,400
				4	499.84				
					611.34	602.64	.995	147.5	33,400
					602.64				

Run No.	Press psia	Inlet Temp OF	Outlet Temp OF	Test Sta. No.	Test Sec Temp OF	T _s	x	h	n/A
16	187	3762	381	1	406.9	383.84	.954	4380	33,400
				2	389.9				
					418.4	396.54	.968	1645	33,200
				3	402.6				
					564.4	522.14	.982	229	33,400
				4	528.2				
					617.4	601.34	.996	148	33,500
					607.4				
17	186	375.75	380	1	403.9	383.84	.953	4130	33,600
				2	389.9				
					417.7	391.54	.968	2110	33,500
				3	397.6				
					530.4	479.84	.983	320	33,600
				4	485.9				
					611.4	598.84	.995	149.5	33,800
					604.9				
18	187	376.2	380.5	1	404.4	384.84	.953	380	33,600
				2	390.9				
					415.9	394.74	.968	1895	33,700
				3	400.4				
					568.7	522.14	.982	229	33,600
					528.2				
				4	618.4	603.84	.996	149	33,800
					609.9				
19	187	376.2	389	1	409.9	384.7	.954	4290	36,400
				2	390.6				
					424.4	402.87	.968	1369	36,400
				3	408.4				
					617.4	583.87	.984	176.2	36,400
				4	589.4				
					637.7	635.37	Supht	--	36,400
					651.9				

Run No.	Press Psia	Inlet Temp of	Outlet Temp of	Test Sta. No.	Test Sec Temp of	T _s	x	h	Q/A
20	187	376.2	389	1	410.4	383.5	.954	5000	36,400
				2	390.0				
				3	426.9	400.5	.968	1495	36,400
				4	407.0				
21	188	376.6	376.6	1	611.9	561.5	.983	197	36,400
				2	568.0				
				3	656.7	631.5		--	36,400
				4	638.0				
22	188	376.6	376.6	1	408.4	386	.953	3395	31,800
				2	391.6				
				3	418.4	395.3	.967	1705	31,800
				4	400.9				
23	188	376.6	376.6	1	534.4	483	.979	297.5	31,600
				2	488.6				
				3	603.4	589.3	.993	150	31,800
				4	594.9				
24	188	376.6	376.6	1	408.9	387.3	.953	2995	31,800
				2	392.9				
				3	418.1	398.3	.966	1475	31,800
				4	403.9				
25	188	376.6	376.6	1	534.9	486.3	.980	291	31,800
				2	491.9				
				3	603.7	589.8	.993	149.5	31,800
				4	595.4				
26	188	376.6	376.6	1	408.4	385.74	.953	3320	30,250
				2	391.6				
				3	416.4	393.54	.965	1777	30,000
				4	399.4				
27	188	376.6	376.6	1	494.4	434.04	.978	527	30,250
				2	439.9				
				3	588.4	574.04	.992	153.5	30,250
				4	579.90				

Run No.	Press psia	Inlet Temp of	Outlet Temp of	Test Sta. No.	Test Sec Temp of	T _s	x	h	Q/A
24	188	376.6	376.6	1	406.9	385.04	.953	3620	30,500
				2	390.9				
					414.9				
				3	398.9	393.04	.965	1851	30,350
25	188	376.6	376.6	4	483.7	431.04	.978	556	30,250
					436.9				
					586.4				
				1	578.7	572.84	.992	154	30,250
					405.9				
				2	385.9	385.9	.953	3290	30,500
					414.4				
				3	393.6	393.6	.965	1781	30,100
26	207	384.7	391		493.7				
				4	439.1	439.1	.978	483	30,250
					587.9				
					573.1	573.1	.992	153.8	30,250
				1	426.2	396.04	.948	3620	41,300
					403.5				
				2	474.2	442.54	.963	679	40,900
27	188	376.6	376.6	3	450.0	614.84	.979	171	41,500
					653.5				
					622.3				
				4	681.2	662.94	.995	150	41,550
28	207	384.7	391		670.4				

Mass Flow Rate 200 lbs/hr.

1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80	81	82	83	84	85	86	87	88	89	90	91	92	93	94	95	96	97	98	99	100
1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80	81	82	83	84	85	86	87	88	89	90	91	92	93	94	95	96	97	98	99	100
1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80	81	82	83	84	85	86	87	88	89	90	91	92	93	94	95	96	97	98	99	100
1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80	81	82	83	84	85	86	87	88	89	90	91	92	93	94	95	96	97	98	99	100

1. The first part of the document is a list of names and addresses. The names are written in a cursive script, and the addresses are written in a more formal, printed script. The list is organized into two columns, with names on the left and addresses on the right.

2. The second part of the document is a list of names and addresses. The names are written in a cursive script, and the addresses are written in a more formal, printed script. The list is organized into two columns, with names on the left and addresses on the right.

NAME	ADDRESS	CITY	STATE	ZIP
JOHN DOE	123 MAIN ST	NEW YORK	NY	10001
JANE SMITH	456 BROADWAY	NEW YORK	NY	10002
ROBERT BROWN	789 PARK AVE	NEW YORK	NY	10003
MARY WHITE	101 E 12th ST	NEW YORK	NY	10004
WILLIAM GREEN	202 W 12th ST	NEW YORK	NY	10005
CHARLES BLACK	303 W 12th ST	NEW YORK	NY	10006
ELIZABETH GRAY	404 W 12th ST	NEW YORK	NY	10007
HENRY KING	505 W 12th ST	NEW YORK	NY	10008
MARGARET HILL	606 W 12th ST	NEW YORK	NY	10009
FRANK LEE	707 W 12th ST	NEW YORK	NY	10010
JOHN PETERSON	808 W 12th ST	NEW YORK	NY	10011
MARY ROSS	909 W 12th ST	NEW YORK	NY	10012
WILLIAM TAYLOR	1010 W 12th ST	NEW YORK	NY	10013
CHARLES WALKER	1111 W 12th ST	NEW YORK	NY	10014
ELIZABETH YOUNG	1212 W 12th ST	NEW YORK	NY	10015
HENRY ZIMMERMAN	1313 W 12th ST	NEW YORK	NY	10016
MARGARET ADAMS	1414 W 12th ST	NEW YORK	NY	10017
FRANK BAKER	1515 W 12th ST	NEW YORK	NY	10018
JOHN CAMPBELL	1616 W 12th ST	NEW YORK	NY	10019
MARY COOPER	1717 W 12th ST	NEW YORK	NY	10020
WILLIAM DAVIS	1818 W 12th ST	NEW YORK	NY	10021
CHARLES EVANS	1919 W 12th ST	NEW YORK	NY	10022
ELIZABETH FOSTER	2020 W 12th ST	NEW YORK	NY	10023
HENRY GIBSON	2121 W 12th ST	NEW YORK	NY	10024
MARGARET HARRIS	2222 W 12th ST	NEW YORK	NY	10025
FRANK JONES	2323 W 12th ST	NEW YORK	NY	10026
JOHN KELLY	2424 W 12th ST	NEW YORK	NY	10027
MARY LAMBERT	2525 W 12th ST	NEW YORK	NY	10028
WILLIAM MASON	2626 W 12th ST	NEW YORK	NY	10029
CHARLES NELSON	2727 W 12th ST	NEW YORK	NY	10030
ELIZABETH O'BRIEN	2828 W 12th ST	NEW YORK	NY	10031
HENRY PETERSON	2929 W 12th ST	NEW YORK	NY	10032
MARGARET QUINN	3030 W 12th ST	NEW YORK	NY	10033
FRANK REED	3131 W 12th ST	NEW YORK	NY	10034
JOHN ROSS	3232 W 12th ST	NEW YORK	NY	10035
MARY SMITH	3333 W 12th ST	NEW YORK	NY	10036
WILLIAM TAYLOR	3434 W 12th ST	NEW YORK	NY	10037
CHARLES WALKER	3535 W 12th ST	NEW YORK	NY	10038
ELIZABETH YOUNG	3636 W 12th ST	NEW YORK	NY	10039
HENRY ZIMMERMAN	3737 W 12th ST	NEW YORK	NY	10040
MARGARET ADAMS	3838 W 12th ST	NEW YORK	NY	10041
FRANK BAKER	3939 W 12th ST	NEW YORK	NY	10042
JOHN CAMPBELL	4040 W 12th ST	NEW YORK	NY	10043
MARY COOPER	4141 W 12th ST	NEW YORK	NY	10044
WILLIAM DAVIS	4242 W 12th ST	NEW YORK	NY	10045
CHARLES EVANS	4343 W 12th ST	NEW YORK	NY	10046
ELIZABETH FOSTER	4444 W 12th ST	NEW YORK	NY	10047
HENRY GIBSON	4545 W 12th ST	NEW YORK	NY	10048
MARGARET HARRIS	4646 W 12th ST	NEW YORK	NY	10049
FRANK JONES	4747 W 12th ST	NEW YORK	NY	10050
JOHN KELLY	4848 W 12th ST	NEW YORK	NY	10051
MARY LAMBERT	4949 W 12th ST	NEW YORK	NY	10052
WILLIAM MASON	5050 W 12th ST	NEW YORK	NY	10053
CHARLES NELSON	5151 W 12th ST	NEW YORK	NY	10054
ELIZABETH O'BRIEN	5252 W 12th ST	NEW YORK	NY	10055
HENRY PETERSON	5353 W 12th ST	NEW YORK	NY	10056
MARGARET QUINN	5454 W 12th ST	NEW YORK	NY	10057
FRANK REED	5555 W 12th ST	NEW YORK	NY	10058
JOHN ROSS	5656 W 12th ST	NEW YORK	NY	10059
MARY SMITH	5757 W 12th ST	NEW YORK	NY	10060
WILLIAM TAYLOR	5858 W 12th ST	NEW YORK	NY	10061
CHARLES WALKER	5959 W 12th ST	NEW YORK	NY	10062
ELIZABETH YOUNG	6060 W 12th ST	NEW YORK	NY	10063
HENRY ZIMMERMAN	6161 W 12th ST	NEW YORK	NY	10064
MARGARET ADAMS	6262 W 12th ST	NEW YORK	NY	10065
FRANK BAKER	6363 W 12th ST	NEW YORK	NY	10066
JOHN CAMPBELL	6464 W 12th ST	NEW YORK	NY	10067
MARY COOPER	6565 W 12th ST	NEW YORK	NY	10068
WILLIAM DAVIS	6666 W 12th ST	NEW YORK	NY	10069
CHARLES EVANS	6767 W 12th ST	NEW YORK	NY	10070
ELIZABETH FOSTER	6868 W 12th ST	NEW YORK	NY	10071
HENRY GIBSON	6969 W 12th ST	NEW YORK	NY	10072
MARGARET HARRIS	7070 W 12th ST	NEW YORK	NY	10073
FRANK JONES	7171 W 12th ST	NEW YORK	NY	10074
JOHN KELLY	7272 W 12th ST	NEW YORK	NY	10075
MARY LAMBERT	7373 W 12th ST	NEW YORK	NY	10076
WILLIAM MASON	7474 W 12th ST	NEW YORK	NY	10077
CHARLES NELSON	7575 W 12th ST	NEW YORK	NY	10078
ELIZABETH O'BRIEN	7676 W 12th ST	NEW YORK	NY	10079
HENRY PETERSON	7777 W 12th ST	NEW YORK	NY	10080
MARGARET QUINN	7878 W 12th ST	NEW YORK	NY	10081
FRANK REED	7979 W 12th ST	NEW YORK	NY	10082
JOHN ROSS	8080 W 12th ST	NEW YORK	NY	10083
MARY SMITH	8181 W 12th ST	NEW YORK	NY	10084
WILLIAM TAYLOR	8282 W 12th ST	NEW YORK	NY	10085
CHARLES WALKER	8383 W 12th ST	NEW YORK	NY	10086
ELIZABETH YOUNG	8484 W 12th ST	NEW YORK	NY	10087
HENRY ZIMMERMAN	8585 W 12th ST	NEW YORK	NY	10088
MARGARET ADAMS	8686 W 12th ST	NEW YORK	NY	10089
FRANK BAKER	8787 W 12th ST	NEW YORK	NY	10090
JOHN CAMPBELL	8888 W 12th ST	NEW YORK	NY	10091
MARY COOPER	8989 W 12th ST	NEW YORK	NY	10092
WILLIAM DAVIS	9090 W 12th ST	NEW YORK	NY	10093
CHARLES EVANS	9191 W 12th ST	NEW YORK	NY	10094
ELIZABETH FOSTER	9292 W 12th ST	NEW YORK	NY	10095
HENRY GIBSON	9393 W 12th ST	NEW YORK	NY	10096
MARGARET HARRIS	9494 W 12th ST	NEW YORK	NY	10097
FRANK JONES	9595 W 12th ST	NEW YORK	NY	10098
JOHN KELLY	9696 W 12th ST	NEW YORK	NY	10099
MARY LAMBERT	9797 W 12th ST	NEW YORK	NY	10100
WILLIAM MASON	9898 W 12th ST	NEW YORK	NY	10101
CHARLES NELSON	9999 W 12th ST	NEW YORK	NY	10102

Run No.	Press psia	Inlet Temp °F	Outlet Temp °F	Test Sta.	Sec No.	Test Temp °F	T _s	x	h	Q/A
27	207	384.7	393		1	427.7	397.04	.998	3490	42,400
					2	404.5				
					3	487.7				
					4	456.8				
					1	653.2	449.34	.964	649	42,400
					2	622.8				
					3	679.9				
					4	669.8				
28	207	384.7	395		1	427.7	615.34	.980	181.5	42,400
					2	405.5				
					3	490.2				
					4	458.5				
					1	656.5	662.34	.996	151.3	42,400
					2	626.3				
					3	681.2				
					4	672.5				
29	207	384.7	396		1	428.5	665	1.00	151.6	42,500
					2	405.9				
					3	478.2				
					4	455.8				
					1	667.2	447.92	.967	720	44,000
					2	624.8				
					3	696.2				
					4	685.2				
30	207	389.7	397		1	428.5	677.32	Supht	--	44,200
					2	405.5				
					3	470.2				
					4	449.5				
					1	664.8	397.7	.949	333.9	43,500
					2	629.5				
					3	694.2				
					4	683.0				
										44,300
										43,200
										43,500

Run No.	Press psia	Inlet Temp of	Outlet Temp of	Test Sta. No.	Test Sec Temp of	T _s	x	h	Q/A
31	207	384.7	384.7	1	418.9	395.67	.948	3330	36,600
				2	402.2	409.47	.963	1505	37,350
				3	416.0				
				4	431.7				
32	207	384.7	384.7	1	538.5	471.97	.978	422	36,600
				2	478.5	606.47	.992	168.5	36,600
				3	624.2	395.67	.948	3320	36,500
				4	613.0	413.97	.963	1260	36,950
33	207	384.7	384.7	1	420.2	541.57	.978	231	36,200
				2	402.2	611.97	.992	160.5	36,500
				3	435.7	395.27	.948	3420	36,300
				4	420.5	413.47	.963	1280	36,800
34	207	384.7	384.7	1	584.2	550.27	.977	217.5	36,000
				2	548.1	613.97	.990	157.1	36,000
				3	628.9	394.0	.948	3820	35,800
				4	618.5	413.0	.962	1290	36,500
				1	419.2	529.3	.976	247	35,800
				2	401.8	608.0	.990	161.5	36,100
				3	440.2				
				4	420.0				
				1	588.0				
				2	556.8				
				3	628.9				
				4	620.5				
				1	418.9				
				2	400.5				
				3	433.9				
				4	419.5				
				1	575.5				
				2	535.8				
				3	625.2				
				4	614.5				

Run No.	Press psia	Inlet Temp	Outlet Temp	Test Sta.	Sec No.	Test Temp	sec of Temp	T _s	x	h	Q/A
35	206	384.3	384.3	1		417.2		393.94	.948	3450	33,350
						400.0					
						429.2		409.44	.960	1339	33,550
						415.5					
36	207	384.7	384.7	1		525.5		479.24	.972	350	33,350
						485.3					
						599.7		584.74	.985	166.1	33,350
						590.8					
						417.2		394.44	.948	3450	33,600
						400.5					
						428.5		405.44	.960	1630	33,800
						411.5					
37	212	387.1	391	1		525.5		482.04	.972	343	33,350
						488.1					
						601.7		587.74	.986	166.7	33,600
						593.8					
38	212	387.7	391	2		432.2		400.5	.947	3620	47,400
						409.0					
						468.2		424.5	.961	1295	47,300
						433.0					
						659.8		617.8	.978	210	47,400
						626.3					
						686.9		678.3	.994	166.5	47,400
						686.8					
38	212	387.7	391	1		431.2		399.51	.946	3810	47,400
						408.0					
						445.7		411.01	.961	1875	47,300
						419.5					
				3		661.5		618.81	.977	204.5	47,400
						627.3					
						687.5		668.01	.991	168.5	47,400
						676.5					

Run No.	Press psia	Inlet Temp of	Outlet Temp of	Test Sta. No.	Test Sec Temp of	T _s	x	h	Q/A
39	212	387.7	391	1	429.7	398.01	.946	4340	47,400
				2	406.5				
				3	445.2				
				4	417.5				
40	212	387.7	392	1	674.8	655.31	.977	177	47,400
				2	663.8				
				3	697.2				
				4	687.5				
41	212	387.1	392	1	428.2	398.01	.946	4370	47,950
				2	406.5				
				3	442.2				
				4	418.9				
42	212	387.1	392	1	651.5	410.41	.961	2050	47,950
				2	615.1				
				3	684.5				
				4	672.5				
				1	432.7	398.71	.947	4080	47,400
				2	407.2				
				3	442.7				
				4	423.2				
				1	648.3	601.61	.978	223	47,400
				2	610.1				
				3	687.5				
				4	676.5				
				1	430.5	668.01	.994	168.5	47,400
				2	405.0				
				3	445.5				
				4	417.0				
				1	649.5	396.5	.946	5050	47,400
				2	610.1				
				3	684.2				
				4	674.2				

Run No.	Press psia	Inlet Temp of	Outlet Temp of	Test Sta. No.	Test Sec Temp of	T _s	x	h	Q/A
43	212	387.1	387.1	1	426.7	397.29	.947	4500	45,800
				2	405.5	400.59	.961	3400	45,600
				3	432.2	501.09	.978	402	45,600
				4	408.8	636.99	.989	183.8	45,400
44	212	387.1	387.1	1	577.0	395.79	.947	5270	45,800
				2	509.3	399.79	.962	3600	45,800
				3	651.2	562.29	.978	260	45,800
				4	645.2	645.59	.989	176.5	45,600
45	212	387.1	387.1	1	422.2	397.17	.946	4240	43,700
				2	404.0	402.17	.961	2880	43,500
				3	433.5	499.97	.976	390	43,700
				4	408.0	626.17	.991	183	43,700
46	212	387.1	387.1	1	605.0	398.17	.945	3960	43,700
				2	570.5	403.67	.958	2620	43,600
				3	661.2	429.47	.973	1035	43,700
				4	653.8	626.97	.988	182.5	43,700

TABLE II
THERMOCOUPLE ERROR - °F

Press psia	1	2	3	4	5	6	7	8	9	10	11	12
199.5	-1.2	-1.5	-1.5	-10.8	-1.2	-6.5	-1.2	-4.5	-8.2	-4.8	-8.8	-8.2
177.5	-1.4	-1.4	-1.4	-8.9	-0.4	-4.9	-0.9	-3.9	-7.1	-4.4	-7.7	-6.4
157	-0.3	-0.8	-0.8	-8.3	-0.3	-4.8	-0.2	-2.0	-6.3	-3.6	-7.3	-6.3
128	-0.5	-0.5	-0.8	-8.0	-0.0	-9.5	-0.5	-5.0	-5.5	-3.5	-6.2	-5.5
100	-0.6	-0.6	-0.9	-5.9	-0.6	-2.4	-0.2	-2.4	-5.9	-3.9	-6.2	-5.6
75	-0.0	-0.5	-1.0	-5.0	-0.5	-2.3	-0.5	-1.0	-5.3	-2.3	-5.3	-4.7
49	-0.6	-0.6	-1.1	-5.1	-1.1	-2.1	-0.6	-2.6	-5.9	-3.9	-5.9	-4.6
35.6	-1.9	-2.4	-2.4	-4.1	-1.9	-3.7	-1.4	-1.9	-6.7	-3.7	-6.1	-5.4
19.0	-2.0	-1.5	-2.0	-3.0	-1.5	-2.0	-1.5	-1.5	-6.0	-4.3	-5.7	-5.3

TABLE III

CALCULATED TEMPERATURE DROP FOR VARIOUS POWER INPUTS

Power Input per Coil in Kilowatts	Temperature Difference Between Surface Thermo- couple and Inside Surface	Temperature Dif- ference Between Depth Couple and Inside Surface
300	11.94	2.80
350	13.92	3.26
400	15.91	3.73
450	17.90	4.20
500	19.92	4.66
550	21.9	5.13
600	23.9	5.60
650	25.9	6.06
700	27.9	6.53
750	29.8	u.00
800	31.8	u.46
850	33.8	7.92
900	35.8	8.40
950	37.8	8.86
1000	39.8	9.32

Year	Month	Day	Time	Location	Remarks
1900	Jan	1	10:00	San Francisco	Arrived
1900	Jan	2	10:00	San Francisco	Departed
1900	Jan	3	10:00	San Francisco	Arrived
1900	Jan	4	10:00	San Francisco	Departed
1900	Jan	5	10:00	San Francisco	Arrived
1900	Jan	6	10:00	San Francisco	Departed
1900	Jan	7	10:00	San Francisco	Arrived
1900	Jan	8	10:00	San Francisco	Departed
1900	Jan	9	10:00	San Francisco	Arrived
1900	Jan	10	10:00	San Francisco	Departed
1900	Jan	11	10:00	San Francisco	Arrived
1900	Jan	12	10:00	San Francisco	Departed
1900	Jan	13	10:00	San Francisco	Arrived
1900	Jan	14	10:00	San Francisco	Departed
1900	Jan	15	10:00	San Francisco	Arrived
1900	Jan	16	10:00	San Francisco	Departed
1900	Jan	17	10:00	San Francisco	Arrived
1900	Jan	18	10:00	San Francisco	Departed
1900	Jan	19	10:00	San Francisco	Arrived
1900	Jan	20	10:00	San Francisco	Departed
1900	Jan	21	10:00	San Francisco	Arrived
1900	Jan	22	10:00	San Francisco	Departed
1900	Jan	23	10:00	San Francisco	Arrived
1900	Jan	24	10:00	San Francisco	Departed
1900	Jan	25	10:00	San Francisco	Arrived
1900	Jan	26	10:00	San Francisco	Departed
1900	Jan	27	10:00	San Francisco	Arrived
1900	Jan	28	10:00	San Francisco	Departed
1900	Jan	29	10:00	San Francisco	Arrived
1900	Jan	30	10:00	San Francisco	Departed
1900	Jan	31	10:00	San Francisco	Arrived



JUL 2
 SEP 8
 OCT 4
 NOV 1
 DEC 3
 FEB 23
 MAR 9
 MAR 25

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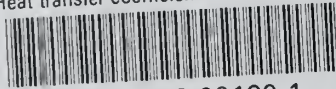
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 steam water mixtures.

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